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(21) International Application Number: PCT/CA92/00292 (22) International Filing Date: 10 July 1992 (10.07.92) (30) Priority data: 727,893                      10 July 1991 (10.07.91)                      US (60) Parent Application or Grant (63) Related by Continuation US    727,893 (CIP) Filed on                                      10 July 1991 (10.07.91) (71) Applicant (for all designated States except US): CANADIAN RUBBER & STEEL LTD. [CA/CA]; 7544 Vantage Place, Tilbury Industrial Park, Delta, British Columbia V4G 1A5 (CA).		(72) Inventor; and (75) Inventor/Applicant (for US only) : DE WAAL, Hendricus, Gerardus [CA/CA]; 5661 125A Street, Surrey, British Columbia V3X 3G8 (CA). (74) Agents: WIGGS, Blake, R. et al.; Barrigar & Oyen, 480 - 601 West Cordova Street, Vancouver, British Columbia V6B 1G1 (CA). (81) Designated States: AT, AU, BB, BG, BR, CH, CS, DE, DK, ES, FI, GB, HU, JP, KP, KR, LK, LU, MG, MN, MW, NL, NO, PL, RO, RU, SD, SE, US, European patent (AT, BE, CH, DE, DK, ES, FR, GB, GR, IT, LU, MC, NL, SE), OAPI patent (BF, BJ, CF, CG, CI, CM, GA, GN, ML, MR, SN, TD, TG). Published With international search report.	
(54) Title: SELF-CLOSING CLAMPING APPARATUS			
(57) Abstract <p>A clamping apparatus (10) for lockingly engaging and supporting an elongate body (11), such as a pipeline span extending above an irregular sea bed. The clamping apparatus (10) includes a saddle (12) shaped to overlie the pipeline span to be supported and an anchor frame (13) straddling the saddle (12) and pivotably coupled thereto. The saddle (12) includes clamps (20a, 20b) which automatically swing to a closed position when the saddle (12) is seated on the elongate body (11). The clamps (20a, 20b) may be releasably locked in the closed position. The anchor frame (13) has extensible legs (40) for supporting the elongate body (11) on an underlying support surface, such as a sandy sea bed. A toolpost (105) may be removably connected to the clamping apparatus for remotely controlling operation of the locking and support leg extension mechanisms.</p>			

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SELF-CLOSING CLAMPING APPARATUSField of the Invention

5           This application relates to a clamping apparatus  
for engaging and supporting a length of conduit, such as an  
underwater pipeline. More particularly, this application  
pertains to a self-closing clamping apparatus which lock-  
10           ingly engages a conduit span when lowered thereon from  
above and which includes extensible legs for supporting the  
conduit span above a support surface, such as the ocean  
floor.

Background of the Invention

15           When conduits, such as gas pipelines, are in-  
stalled underwater they often can not lie flat on the ocean  
floor. Rather, depressions or risers on the ocean floor  
may result in unsupported pipeline spans of considerable  
20           length. Some means must be provided to support the pipe-  
line at such locations in order to prevent dynamic vortex-  
induced oscillations of the pipeline and subsequent risk of  
fatigue failure. For gas pipelines, the risk of stress  
fatigue is particularly acute when the pipeline is in-  
25           itially hydrotested.

            Conventionally, cement grout is pumped from a  
surface vessel into bags placed underneath the pipeline  
span which is to be supported. This is an extremely costly  
30           and time-consuming procedure, especially in areas having  
undulating terrain. If the sea bed slope is particularly  
steep or the soil soft, special care must be taken to  
prevent the grout support bags from slipping out of place  
before the grout has set. In such instances grout bags  
35           must be filled gradually via several successive pours of  
grout material. Even if precautions are taken, failure  
rates on the order of 50% may occur.

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The time required to position and fill a single, multi-layered grout bag in deep water is typically on the order of 24-36 hours. Installation time may be lengthened significantly in the event of inclement weather, since the surface vessel must be maintained in a relatively station-  
5 ary position during the entire procedure. Another major drawback of the conventional procedure is that a large amount of equipment and manpower is required to position and fill the grout bags.

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It is known in the prior art to provide mechanical devices for engaging and elevating conduits, such as underwater pipelines. For example, United States Patent No. 4,494,893 which issued to Migliavacca on 22 January,  
15 1985 discloses an adjustable apparatus for supporting an underwater pipe laid at great depth over a depression in the sea bed. The Migliavacca apparatus is best suited for supporting pipelines extending generally horizontally relative to the underlying sea bed, or at a shallow incline  
20 relative to the sea bed. However, pipeline spans may sometimes extend at a significant incline relative to the sea bed, especially in areas having very undulating terrain. The Migliavacca apparatus is unsuitable in these applications due to the integral connection between the  
25 clamps and the upper portions of the support legs. Since the Migliavacca clamps will only function as intended when the apparatus hangs in a vertical orientation, the apparatus may not be used to engage vertically extending lengths of conduit, such as upstanding oil derricks and the like.

30

Accordingly, the need has arisen for a clamping apparatus which may be lowered onto a length of conduit from above and which includes means for automatically engaging the conduit and for securely anchoring the conduit  
35 above a support surface, such as an underlying sea bed, even in cases when the conduit extends at a significant angle relative to the support surface.

Summary of the Invention

In accordance with the invention, a self-closing clamping apparatus for engaging and supporting an elongate body, such as a length of conduit, is provided. The clamping apparatus includes a saddle shaped to conform to an exterior surface of the elongate body, the saddle having opposed side portions defining a compartment therebetween for receiving the elongate body. Clamping means hingedly connected to the saddle are swingable between an open position partially extending within the compartment and a closed position clampingly engaging a portion of the elongate body exterior surface. The clamping means automatically swings from the open position to the closed position when the saddle is seated on the elongate body.

15

Preferably the clamping means includes a pair of clamps, each clamp being hingedly connected to one of the saddle side portions. The saddle side portions are slotted to permit swinging movement of the clamps between the open and closed positions through the saddle side portions.

20

Advantageously, each of the clamps includes a clamping surface shaped to conform to the elongate body exterior surface, the clamping surface having first and second ends. When the clamping apparatus is in the open position, the clamp first ends extend outwardly of the saddle side portions and the clamp second ends extend within the compartment. Placement of the elongate body into the compartment as the saddle is seated on the elongate body forces the clamping surface second ends outwardly away from one another, thereby swinging the clamping surface first ends inwardly toward one another.

30

Preferably, the clamping apparatus also includes biasing means for urging the clamping means to the open position. The biasing means may include a tensile member connectible between the clamping surface first ends.

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Locking means may also be provided for releasibly locking the clamping means in the closed position. Preferably, the locking means fastens the clamping surface first ends together. In one embodiment, the locking means may include a bolt rotatably coupled to one of the clamps and swingable therewith. In an alternative embodiment, the locking means may include a housing mounted on the saddle; a bolt rotatable within the housing; and linkage means for operatively connecting the bolt and the clamping means.

The clamping apparatus may be adapted for engaging a cylindrical conduit elevated above a support surface. The saddle may include a semi-cylindrical upper surface extending between the saddle side portions for overlying an upper portion of the conduit. The clamps may include an arcuate clamping surface for engaging a lower portion of the conduit in the closed position.

Anchor means coupled to the saddle may also be provided for anchoring the clamping apparatus on a support surface. The anchor means may include a pair of support legs, each support leg being pivotably connected to one of the saddle side portions. The support legs are adapted for extending between the elevated conduit and the support surface. Since the centre of gravity of the support legs is below the point of pivotal connection of said support legs to said saddle, the support legs extend in a substantially vertical plane irrespective of the angle of inclination of the saddle relative to the support surface. Preferably, the support legs are angled outwardly to provide lateral support for the elevated conduit. The upper portions of the support legs may be connected by a transverse crosspiece, the support legs and the crosspiece together defining a swingable yoke straddling the saddle.

- 5 -

A mounting post may be provided on the transverse crosspiece for removably receiving a toolpost. The toolpost preferably includes a suspension cable for retrieving the toolpost from an underwater location, and remotely  
5 controllable actuating means mounted on the toolpost for releasibly engaging and activating the alternative embodiment of the locking means referred to above. The actuating means preferably includes a tool holder for receiving the lock housing and for coupling a hydraulic wrench to the  
10 bolt rotatable within the housing.

Advantageously, the support legs include telescopic shafts adjustable between retracted and extended positions. Elongated screws rotatable within respective  
15 shafts may be provided for adjusting the shafts between the retracted and deployed positions. Preferably, the toolpost actuating means further includes a tool holder for releasibly coupling a hydraulic wrench to an upper portion of the elongated screws.

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Flared boots may be mounted at the lower end of the support legs for securely engaging the support surface and for spreading the load of the conduit on the support surface.

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In an alternative embodiment, the clamping apparatus is not self-closing. Rather, the clamping apparatus may broadly comprise a saddle shaped to conform to an exterior surface of an elongate body; clamping means  
30 mounted on the saddle for engaging the elongate body; and anchor means pivotably coupled to the saddle for anchoring the elongate body on a support surface.

A method for supporting an elongate body in an elevated position above a support surface is also disclosed  
35 which includes the steps of (a) providing a clamping apparatus of the type comprising a frame, clamping means

- 6 -

hingedly coupled to the frame for engaging the elongate body, the clamping means being adjustable between an open position and a closed position clasping the elongate body, locking means for releasibly locking the clamping means in the closed position, and a pair of extensible support legs pivotably coupled to the frame for extending from the frame to the support surface; (b) determining the preferred elevation of the elongate body above the support surface; (c) suspending the clamping apparatus above the elongate body to be supported; (d) lowering the clamping apparatus onto the elongate body until the clamping means securely engages the elongate body; (e) actuating the locking means to securely lock the clamping means in the closed position; (f) extending one of the support legs until said leg contacts the support surface, thereby resulting in upward deflection of the elongate body; (g) extending the other of the support legs until said other leg contacts the support surface, thereby resulting in further upward deflection of the elongate body; and (i) repeatedly extending the support legs until the elongate body is raised to its preferred elevation as determined in step (a).

#### Brief Description of the Drawings

In the accompanying drawings which illustrate various embodiments of the invention,

Figure 1 is a fragmented, isometric view of the clamping apparatus and attached toolpost in a deployed configuration seated on a conduit span;

Figure 2 is a side elevational view of the clamping apparatus and toolpost of Figure 1 with the support legs retracted;

Figure 3 is an end elevational view of the clamping apparatus of Figure 1 showing the toolpost ex-



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ploded from the upper end thereof and showing the support legs partially extended in phantom outline;

Figure 4 is an enlarged, fragmented, isometric  
5 view of the clamping apparatus saddle in a deployed configuration seated on a conduit span;

Figure 5 is an enlarged, fragmented end view of  
the clamping apparatus saddle about to be seated on a  
10 conduit span, showing the clamping assembly in an open configuration;

Figure 6 is an end view of the clamping apparatus saddle of Figure 5 seated on a conduit span, showing  
15 the clamping assembly in a closed and locked configuration;

Figure 7 is a fragmented, top plan view of the  
clamping apparatus and attached toolpost, showing the  
clamping assembly in a closed position;

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Figure 8 is an enlarged, bottom plan view of the  
clamping apparatus, showing the clamping assembly in a  
closed configuration and showing one embodiment of the  
locking assembly in a locked configuration;

25

Figure 9 is an end view of the clamping apparatus  
saddle about to be seated on a conduit span showing the  
clamping assembly in an open position and illustrating an  
alternative embodiment of the locking assembly;

30

Figure 10 is an end view of the clamping apparatus saddle of Figure 9 seated on a conduit span showing  
the clamping assembly in a closed position;

35

Figure 11 is an end view of the clamping apparatus saddle of Figure 10 showing the alternative locking  
assembly in a locked configuration;

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Figure 12 is a side elevational view of the clamping apparatus of Figure 10 seated on an inclined conduit;

5

Figure 13 is an enlarged, fragmented sectional view of one of the extensible support legs;

Figure 14 is an end elevational view of the clamping apparatus saddle about to be seated on a conduit span showing the clamping assembly in an open position and illustrating a further alternative embodiment of the clamping assembly having truncated clamp halves;

Figure 15 is an end elevational view of a lock activating assembly mountable on the toolpost for engaging the locking assembly of Figures 9-12 and 14;

Figure 16 is a side elevational view of the lock activating assembly of Figure 15;

Figure 17(a) is a top plan view of the cylinder housings of the lock activating assembly;

Figure 17(b) is a side elevational view of the cylinder housings of Figure 17(a);

Figure 18 is an enlarged, partially fragmented longitudinal sectional view of a dual piston, dual rod hydraulic cylinder mounted within one of the cylinder housings;

Figure 19(a) is a top plan view of the hydraulic cylinder of Figure 18;

35

Figure 19(b) is a side elevational view of the hydraulic cylinder of Figure 18;

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Figure 19(c) is an end elevational view of the hydraulic cylinder of Figure 18;

5        Figure 20(a) is a top plan view of one of the cylinder housings of Figure 17;

Figure 20(b) is a side elevational view of the cylinder housing of Figure 20(a);

10

Figure 20(c) is a bottom plan view of the cylinder housing of Figure 20(a);

15        Figure 21 is a fragmented, end elevational view of a cylinder housing showing a pair of external locking lugs mounted thereon;

Figure 22 is an end elevational view of one of the brackets for mounting on a locking assembly housing;

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Figure 23 is an end elevational view of the tool holder slidably coupled to a cylinder housing;

25        Figure 24 is a top plan view of the tool holder of Figure 23; and

Figure 25 is a top plan view of a plate for engaging one of the ends of the lock assembly housing.

30        Detailed Description of the Preferred Embodiment

Figure 1 depicts a clamping apparatus 10 for clampingly engaging and supporting an elongate body, such as a length of cylindrical conduit 11 (typically a gas pipeline) extending above a support surface, such as the ocean floor.

- 10 -

Clamping apparatus 10 includes an elongated saddle 12 shaped to overlies a length of conduit 11, and an anchor frame 13 straddling saddle 12 and pivotally coupled thereto. As discussed in further detail below, anchor frame 13 is swingable relative to saddle 12 and includes extensible support legs generally designated 40 for anchoring clamping apparatus 10 on the underlying support surface.

With reference to Figures 4-6, saddle 12 consists of a pair of opposed sidewalls 14 integrally connected by a semi-cylindrical upper wall 16. Sidewalls 14 and upper wall 16 together define an open-bottomed compartment 15 for receiving a length of conduit 11 (Figure 5). Sidewalls 14 are preferably planar and, when clasped about conduit 11 as hereinafter explained, are inclined outwardly at an angle of approximately 15° relative to the vertical. Stiffeners 17 may be fitted around either outer end of saddle 12 for structural support.

At least one clamp 20 (Figure 2) is hingedly connected to saddle 12. In the preferred embodiment, two spaced-apart, self-closing clamps 20 are provided. Each clamp 20 includes clamp halves 20(a) and 20(b) which are hingedly connected to opposed saddle sidewalls 14 (Figure 5). Saddle sidewalls 14 have cut-out slots 18 to permit free swinging movement of clamp halves 20(a) and 20(b) relative to saddle 12.

The number of clamps 20 mounted on saddle 12 may vary depending on the clamping force required for a particular application. For example, in some applications, saddle 12 could be elongated to enable mounting of three, four or more clamps 20. Preferably, the length of saddle 12 should be at least such that the surface area of saddle upper wall 16 in contact with conduit 11 is equal to or greater than the effective combined surface area of clamps

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20 in contact with conduit 11 when clamping apparatus 10 is deployed. Of course, the clamping surface area in contact with conduit 11 may vary depending on the loads required to be transmitted through saddle 12.

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In alternative embodiments, saddle 12 may consist of a framework which is not shaped to overlies conduit 11 and which is not seated directly on conduit 11 in the deployed position. In such alternative embodiments saddle 12 need not be elongated. As described in detail below, the primary function of saddle 12 is to support clamps 20, which positively clasp conduit 11 in the closed position, and to support legs 40, which may be extended from saddle 12 to an underlying support surface, such as the ocean floor.

As shown best in Figure 5, each clamp half 20(a), 20(b) has a first end 28, a second end 30 and an arcuate clamping surface 31 for engaging conduit 11 over an arc extending approximately 145 degrees between ends 28, 30. Clamp halves 20(a), 20(b) are connected part way between ends 28, 30 to a respective saddle sidewall 14 by a hinge generally designated 22 (Figure 2). As shown in Figure 4, hinge 22 includes a tubular hinge sleeve 24 mounted on each clamp half 20(a), 20(b) and alignable with corresponding tubular hinge sleeves 26 mounted on opposite sides of saddle slots 18 (Figure 4). Hinge 22 also includes a hinge pin 25 (Figures 5-6) insertable through hinge sleeves 24, 26 to securely couple clamp halves 20(a) and 20(b) to respective saddle sidewalls 14. A stop bar 27 is fixed to clamp hinge sleeve 24 to limit rotation of sleeve 24 relative to sleeves 26. In an alternative embodiment, saddle hinge sleeves 26 may be omitted and shims may be positioned under the ends of hinge pin 25 to compensate for the offset resulting from the wall thickness of hinge sleeve 24. In a further alternative embodiment the plate thickness of saddle 12 adjacent clamps 20 may be increased.

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Clamp halves 20(a), 20(b) are swingable about their respective hinges 22 to an open position shown in Figure 5 in which second ends 30 extend within conduit receiving compartment 15, and first ends 28 extend outwardly of saddle sidewalls 14. When saddle 12 is lowered on to a length of conduit 11, clamps 20(a), 20(b) are automatically carried into the closed position shown in Figure 6. In particular, as second ends 30 contact conduit 11 they swing outwardly away from one another, thereby swinging first ends inwardly and downwardly toward one another through slots 18. In the closed position shown in Figure 6 the arcuate clamping surfaces 31 of each clamp half 20(a), 20(b) clasp respective lower portions of conduit 11.

15

As shown best in Figure 5, clamping surfaces 31 and the inner surface of saddle upper wall 16 contacting conduit 11 may be covered with an lining 34 to cushion conduit 11 and protect against corrosion.

20

In the preferred embodiment of the invention, a biasing means, such as an elastomeric bungee cord 35, is also provided for urging clamps 20(a) and 20(b) toward the open position. As shown best in Figures 5-7, elastomeric cord 35 is fastened at one end to a hasp 150 connected to bolt housing 62 mounted on clamp half 20(a); and, at the opposite end, to a hasp 152 connected to plate 36 which extends from first end 28 of clamp half 20(b). Elastomeric cord 35 is entrained over the arcuate upper wall 16 of saddle 12 to urge first ends 28 upwardly and outwardly toward the open position shown in Figure 5 when saddle 12 is not seated on conduit 11. As shown in Figures 4 and 5, stop bar 27 fixed to clamp hinge sleeve 24 prevents rotation of clamp halves 20(a), 20(b) beyond the preferred open position (in which second ends 30 extend within conduit receiving compartment 15). When saddle 12 is lifted clear of conduit 11, elastomeric cord 35 causes clamp halves

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20(a), 20(b) to swing toward the open position until stop bar 27 contacts saddle upper wall 16 (Figure 5).

Clamping apparatus 10 may also include "guiding means" for guiding a length of conduit 11 into conduit receiving compartment 15. As shown best in Figures 4 and 8, the guiding means preferably comprises triangular flared guide plates 37 which are integrally joined to the lower ends of saddle sidewalls 14 and extend downwardly and outwardly therefrom. Stiffening ribs 19 bridging the central portion of saddle sidewalls 14 and guide plates 37 may be provided for added structural support. Guide plates 37 serve to guide saddle 12 onto conduit 11, such as by causing saddle 12 to shift sideways as it is lowered to ensure that the longitudinal axes of saddle 12 and conduit 11 are in alignment. Guide plates 37 are preferably outwardly inclined at an angle of approximately 27° relative to a vertical axis (or approximately 13° relative to the plane of saddle sidewalls 14).

20

Clamping apparatus 10 also includes "locking means" for releasibly locking each pair of clamp halves 20(a), 20(b) together in the closed position. As shown in Figures 5 and 6, the locking means preferably comprises a captured, externally threaded bolt 60 and an internally threaded block nut 66 which travels to a locking position when bolt 60 is rotated. Bolt 60 is rotatable within bolt housing 62 which is pre-mounted on clamp half 20(a). More particularly, bolt housing 62 is integrally connected to plate 32 which extends from first end 28 of clamp half 20(a). Bolt housing 62 is fixed proximate hinge sleeve 24 by means of a pair of rods 64. In the unlocked position, block nut 66 is positioned at the exposed, free end of bolt 60 remote from housing 62 (Figure 5). A keeper plate 68 is fixed on the bolt free end to prevent detachment of nut 66 from bolt 60.

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Bolt 60 preferably has a threaded stepped portion at the bolt end opposite keeper plate 68 for receiving an internally threaded hexagonal coupling nut 70. After coupling nut 70 has been screwed onto bolt 60 and set for correct clearance, nut 70 is integrally connected to bolt 60, such as by welding. Rotation of bolt 60 is actuated by applying a rotary force to nut 70, for example with the aid of a hydraulic wrench. Nut 70 is separated from bolt housing 62 by a loose-fit cylindrical spacer 72. Spacer 72 abuts against a loose-fit flange 74 which is fastened to the flanged end 76 of bolt housing 62 with bolts 78 to prevent axial travel of bolt 60.

When clamp halves 20(a), 20(b) are swung to the closed position shown in Figure 6, bolt 60 assumes a generally horizontal orientation underneath conduit 11. The first end 28 of mating clamp half 20(b) carries a slotted plate 33 which receives the exposed end of bolt 60. In the closed position, one side of block nut 66 is seated against plate 36 extending from first end 28 of clamp half 20(b); this prevents nut 66 from rotating freely with bolt 60. Hence, when a rotary force is applied to hexagonal nut 70, such as by a hydraulic impact wrench manipulated and activated by an underwater diver or remote underwater vehicle, bolt 60 rotates, causing block nut 66 to travel along the length of bolt 60 toward housing 62 until block nut 66 is seated securely against plate 33, thus locking clamp halves 20(a) and 20(b) together (Figures 6 and 8). The locking mechanism may be released by rotating bolt 60 in the opposite direction.

An alternative embodiment of the locking means is illustrated in Figures 9-12. In this alternative embodiment a locking assembly generally designated 79 is mounted on saddle upper surface 16 immediately above clamps 20. As discussed in further detail below, the primary advantage of this alternative embodiment is that locking assembly 79



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is more readily accessible to divers or underwater remote operated vehicles (ROVs) fitted with hydraulic wrenches. Locking assembly 79 is also preferable if clearances underneath saddle 12 are limited.

5

Alternative locking assembly 79 comprises a unitary shaft 80 having hexagonal end portions 82; and, left hand threaded and right hand threaded portions 84, 86 respectively. Shaft 80 extends within housing 88 which is shaped to overlies saddle upper wall 16, immediately above clamps 20. In this embodiment, stiffeners 17 (Figure 4) are ordinarily omitted, since housing 88 provides sufficient structural support for saddle 12.

15

Nuts 90, 92 are threaded onto shaft threaded portions 84, 86 respectively. Hence, rotation of shaft 80 causes nuts 90, 92 to travel outwardly (i.e. away from one another) toward shaft ends 82; or, inwardly (i.e. toward the centre of shaft 80) depending upon the direction of rotation. Shaft 80 thus functions as a turnbuckle.

20

Each nut 90, 92 is operatively connected to either clamp half 20(a) or 20(b) by means of a pair of linkages 96. Each linkage 96 has an upper end 98 and a lower end 99. As shown in Figure 9, linkage upper end 98 is preferably apertured for receiving cylindrical stubs 100 which are machined on the lateral surfaces of nuts 90, 92. A set collar 97 may be provided for loosely coupling linkage upper ends 98 to stubs 100.

25

Linkage lower end 99 is coupled to a hinge assembly 102 mounted on clamp halves 20(a) or 20(b) as the case may be. Hinge assembly 102 includes a hinge pin 104 which is mounted on clamp halves 20(a) or 20(b) outwardly of hinge 22 by means of connector plates 106. Connector plates 106 are integrally connected to the arcuate outer

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surface of clamp halves 20(a) and 20(b) and are apertured to accommodate hinges 22 and 102.

Set collars 107 or similar fasteners are fitted  
5 on to the ends of hinge pin 104 to loosely couple pin 104 to linkages 96. As shown clearly in Figures 9-11, linkage lower end 99 includes an elongate slot 101 to enable pivoting movement of hinge assembly 102 relative to saddle 12. In particular, hinge pin 104 pivots freely within  
10 linkage slot 101 as clamp halves 20(a) and 20(b) swing between the open position (Figure 9) and the closed position (Figure 10) as saddle 12 is seated on conduit 11.

In the alternative locking assembly 79 described  
15 above, the biasing means preferably consists of a pair of elastomeric cords 35, each fastened at one end to housing 88 above saddle 16 and at the other end to a hasp projecting from the first end 28 of clamp half 20(a) or 20(b) as the case may be. Cords 35 bias clamp halves 20(a) and  
20 20(b) toward the open position wherein the clamp second ends 30 extend within the conduit receiving compartment 15 defined by saddle 12 (Figure 9).

With reference to locking assembly 79 illustrated  
25 in Figures 9-11, clamp halves 20(a) and 20(b) are locked in the closed position by rotating shaft 80, such as by securing a hydraulic impact wrench to one of the shaft hexagonal end portions 82 and actuating the wrench. Depending upon which end 82 of shaft 80 the wrench is  
30 secured to, clockwise or counterclockwise rotation of shaft 80 will cause nuts 90, 92 and attached linkages 96 to travel outwardly toward shaft ends 82. (The end plates of housing 88 supporting shaft hexagonal end portions 82 are preferably labelled to indicate which direction of rota-  
35 tion of shaft 80 will cause outward travel of nuts 90, 92). Outward travel of nuts 90, 92 and attached linkages 96 applies an inwardly directed clamping force to clamp halves

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20(a), 20(b) through hinge assembly 102 and connector plates 106. Clamp halves 20(a) and 20(b) are thus securely locked in the closed position as shown in Figure 11. Since alternative locking assembly 79 causes clamp halves 20(a) and 20(b) to positively embrace conduit 11, it more effectively corrects for improper seating of saddle 12 than the first embodiment of the locking means employing captured bolt 60 and travelling nut 66 referred to above.

10           Clamp halves 20(a) and 20(b) may be returned to the unlocked position by rotating shaft 80 in the opposite direction, thus causing inward travel of nuts 90 and 92 and attached linkages 96. This releases the inwardly directed clamping force acting upon clamp halves 20(a) and 20(b) and  
15           allows hinge pins 104 to pivot upwardly within linkage slots 101 when saddle 12 is lifted clear of conduit 11. As described above, elastomeric cords 35 urge clamp halves 20(a) and 20(b) to swing to the open position when saddle 12 is lifted clear of conduit 11.

20

          In a further alternative embodiment of the invention shown in Figure 14 the second ends 30 of clamp halves 20(a) and 20(b) are truncated so that they do not extend within conduit receiving compartment 15 in the open  
25           position. Accordingly, in this embodiment clamp halves 20(a), 20(b) do not close automatically when saddle 12 is seated on conduit 11. Rather, closure and locking of clamp halves 20(a), 20(b) is achieved by actuating locking assembly 79 after saddle 12 is seated on conduit 11. That  
30           is, rotation of shaft 80 causes outward travel of nuts 90 and 92 and attached linkages 96 which in turn results in transmittal of an inwardly directed clamping force to clamp halves 20(a) and 20(b) through linkages 96, hinge assembly 102 and connector plates 106 as discussed above. This  
35           causes truncated clamp halves 20(a) and 20(b) to swing to the closed position and lockingly engage conduit 11. In this alternative embodiment linkage slots 101 and elasto-

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meric cords 35 are omitted since it is not necessary to allow for swinging movement of clamp halves 20(a) and 20(b) when saddle 12 is seated on conduit 11 (i.e hinge pin 104 is immovably rather than slidably coupled to linkage second ends 99).

The further alternative locking mechanism shown in Figure 14 is particularly suited for applications wherein seating of saddle 12 on conduit 11 does not result in sufficient upwardly directed forces to actuate closure of clamps 20. For example, if buoyancy members are attached to clamping apparatus 10 during deployment, seating of saddle 12 on conduit 11 may not overcome the biasing forces urging clamp halves 20(a), 20(b) to the open position.

As indicated at the outset, clamping apparatus 10 also includes an anchor frame 13 straddling saddle 12 for supporting conduit 11 on an underlying support surface. Frame 13 enables a load to be transmitted to or from conduit 11 through saddle 12.

With reference to Figure 3, anchor frame 13 includes an inverted U-shaped plate 39 and a pair of extensible support legs 40 which are integrally joined to opposite outer edges of plate 39. More particularly, each leg 40 has an elongated square tubular housing 110 (Figure 4) which is welded to the outer lateral edges of plate 39. Plate 39 is preferably about one half inches in thickness and has external ribs 41 to provide enhanced rigidity. As discussed below, ribs 41 are slotted to accommodate trunnions 48 for coupling anchor frame 13 to saddle 12. The undersurface of plate 39 proximate its apex is arcuately curved to conform to the shape of saddle upper wall 16.

As shown in Figure 4, the upper ends of leg member outer housings 110 are joined by a transverse cross-

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piece 42 integrally connected to plate 39. Crosspiece 42 extends above saddle 12 when clamping apparatus 10 is assembled. A post 44 having an aperture 46 formed therein projects upwardly from a central portion of crosspiece 42.

- 5 As discussed in further detail below, post 44 is provided for coupling clamping apparatus 10 to a toolpost generally designated 105 (Figures 1-3).

A tubular cage 113 is integrally connected to the  
10 outer lateral surface of leg housings 110 as shown in Figures 1-3 and 7. Cage 113 is provided to protect saddle 12 during shipping. Additionally, cage 113 provides a rigid structure for manipulating clamping apparatus 10 by underwater divers or underwater remote operated vehicles  
15 (ROVs) during deployment.

Frame leg members 40 are gimbaled to saddle 12 by means of a pair of in-line trunnions 48. As shown best in Figures 4-6, each trunnion 48 includes a cylindrical  
20 shaft 52 extending outwardly of saddle 12 perpendicular to the longitudinal axis of the saddle. Each shaft 52 is integrally connected to a mounting plate 53 located on a central portion of saddle 12 between clamps 20 (Figure 4). During assembly of trunnions 48 a cylindrical bushing 50 is  
25 loosely fitted on each shaft 52 and a retainer ring 56 is then welded to the free end of shaft 52 to prevent detachment of bushing 50.

During fabrication of clamping apparatus 10  
30 slotted ribs 41 are initially welded to plate 39 in two steps to enable trunnions 48 to be connected to anchor frame 13. More particularly, one half of rib 41 is initially welded to plate 39 and then trunnions 48, comprising bushing 50, shaft 52 and retainer ring 56, are fitted  
35 into the slot defined by plate 39 and one half of rib 41. The other half of rib 41 is then welded to plate 39 so that each bushing 50 is captured within an aperture defined by

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plate 39 and rib 41. Finally bushing 50 is welded to plate 39 and the adjacent rib 41. Once trunnions 48 are assembled as aforesaid, each shaft 52 can rotate freely relative to its corresponding bushing 50 and hence frame 13  
5 can swing freely relative to saddle 12.

In order for clamping apparatus 10 to work effectively, it is important that trunnions 48 share a common rotational axis. Depending on the application,  
10 trunnions may be offset or non-symmetrically mounted to perform a given function.

Clamping apparatus 10 is designed so that the centre of gravity of anchor frame 13 is below saddle trunnions 48. Accordingly, since frame 13 is freely swingable  
15 relative to saddle 12, extensible support legs 40 will extend substantially vertically irrespective of the angle of inclination of conduit 11 and hence saddle 12 (Figure 12). Clamping apparatus 10 is thus designed to accommodate  
20 varying pipe slopes while maintaining support legs 40 in a vertical orientation.

As should be apparent to someone skilled in the art, other means for pivotably coupling anchor frame 13 to  
25 saddle 12 may be substituted for trunnions 48. For example, a ball and socket joint or universal connection allowing rotation of anchor frame 13 in more than one plane relative to saddle 12 could be provided.

30 As indicated above, ~~anchor frame 13~~ includes extensible support legs 40 for anchoring clamping apparatus 10 on a support surface, such as the ocean floor. With reference to Figure 13, which is a detailed illustration of the assembly of one support leg 40, reference numeral 110  
35 denotes the outer housing which is welded to plate 39 as discussed above. An extensible inner leg 112 is telescopically slidable within housing 110. Both housing 110 and

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leg 112 are constructed of square tubing so they will not rotate relative to one another.

Support leg 40 also includes an externally threaded leg screw 114 which is fixed at its upper end to a transverse plate 115 extending within housing 110. The upper end of leg screw 114 includes a hexagonal end portion 116 for coupling with a hydraulic wrench to actuate rotation of screw 114. Screw 114 is rotatably coupled to plate 115 by means of threaded nut 117 and thrust washer 118. Nut 117 is securely connected to screw 114, such as by machining a bore through nut 117 and screw 114 and inserting a locking pin therethrough.

An internally threaded square-shaped screw nut 120 is provided for coupling screw 114 to inner leg 112. In particular, the periphery of screw nut 120 is integrally connected to leg 112 during assembly of leg member 40 such as by welding. Thus nut 120 and leg 112 move together as a unit.

Since leg screw 114 is fixed at its upper end to housing 110, rotation of screw 114 causes nut 120 and hence inner legs 112 to travel downwardly relative to housing 110. Thus rotation of screw 114 causes slidable telescopic extension of inner leg 112 relative to housing 110. The outer surface of inner leg 112 is preferably marked at regular intervals to provide a visual indication of the extent of leg extension.

30

A bushing 122 is fitted at the lower end of leg screw 114 as shown in Figure 13. Bushing 122 is dimensioned to rotate freely relative to screw 114 and is provided to limit lateral flexing of screw 114 as it is rotated. A stop plate 124 is provided to prevent bushing 122 from sliding free from the lowermost end of screw 114. Upward travel of bushing 122 relative to screw 114 is

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limited by a shoulder 126 formed in screw 114. If shoulder 126 is omitted, bushing 122 has a tendency to travel upwardly relative to screw 114 when inner leg 112 is retracted due to frictional forces.

5

As shown best in Figures 1-3, the lowermost end of each inner leg 112 is provided with a load transferring boot 128 for engaging the underlying support surface, such as a sandy seabed. The design of boot 128 may vary depending upon soil conditions. For example, boot 128 illustrated in the drawings is designed to spread a load onto a soft seabed. A boot 128 having a pointed end with a small mud mat may be more suitable for harder soils. Boots 128 are shaped to maximize the distance therebetween so as not to obstruct deployment of clamping apparatus 10 on conduit 11 (Figure 1). In some applications extra weight is mounted on boots 128 so that the overall centre of gravity of anchor frame 13 is below trunnions 48, thus ensuring that support legs 40 hang vertically as discussed above.

20

In order to limit corrosion of support legs 40 after deployment in salt water, non-ferrous anode bars 130 may be mounted on the outer surface of outer housing 110 and inner leg 112 (Figures 1-3). The anode bar 130 mounted on inner leg 112 is preferably secured to a mounting plate having a outwardly tapered upper end so that it does not interfere with complete retraction of inner leg 112.

Hasps 154 may also be mounted on support legs 40 for attachment to underwater cables.

As should be apparent to someone skilled in the art, other means for anchoring frame 13 on an underlying support surface other than extensible legs 40 may be employed. For example, multiple scissor linkages or telescopic legs which are not screw activated could be substituted.



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As shown best in Figures 1-3, toolpost 105 is provided for use in deploying clamping apparatus 10. Toolpost 105 is designed to slide onto mounting post 44 extending upwardly from frame crosspiece 42. Toolpost 105 includes a pin linkage 109 releasibly insertable in aperture 46 formed in mounting post 44 (Figure 3). Toolpost 105 also includes a pair of tool holders 108 fitted with hydraulic wrenches or the like (not shown) which are mountable on the hexagonal end portions 116 of leg screws 114 projecting from the upper end of leg member housings 110 (Figures 3 and 13). The uppermost end of toolpost 105 has a hasp 111 formed therein for securing a suspension cable. Toolpost 105 may function as a frame for supporting lights, cameras, hydraulic cylinders, hydraulic lines, inclinometers and other accessories required to deploy clamping apparatus 10.

Figures 15 and 16 depict one particular embodiment of the invention wherein toolpost 105 functions as a frame for supporting a lock activating assembly 160. Assembly 160 is designed for remotely controlling operation of the alternative locking assembly 79 described above and depicted in Figures 9-12 and 14. This avoids the need to mount hydraulic impact wrenches on the manipulator arms of the ROV to actuate locking assembly 79, which is a significant advantage in many applications. For example, in the case of shallow pipeline spans, there is often very little clearance between saddle 12 and the underlying sea bed. Accordingly, it is necessary to manoeuvre the ROV close to the sea bed in order to lock clamps 20 in the closed position. As the ROV approaches the sea bed it tends to stir up sand or other surface debris which impairs visibility and makes it difficult for the surface operator to manoeuvre a hydraulic wrench pre-mounted on the ROV onto shaft 80 of locking assembly 79 (Figures 9-11 and 14). By mounting all of the hydraulic impact wrenches directly on

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toolpost 105, the ROV is no longer required to perform any of the screw-activating tasks. That is, tool holders 108, each fitted with a hydraulic wrench or the like, are mounted on toolpost 105 for controlling operation of screw-activated extensible legs 40 (Figures 1 and 3), and lock activating assembly 160 is also mounted on toolpost 105 for releasibly locking clamps 20 in the closed position (Figures 15 and 16), as described in further detail below.

10 In the preferred embodiment as shown in Figure 12, clamping apparatus 10 includes two pairs of clamps 20 and two corresponding locking assemblies 79, each for extending above a corresponding clamp set. Each locking assembly 79 includes a housing 88 and a shaft 80 rotatable within housing 88 (Figures 9-12). Lock activating assembly 160 (Figure 15) is mounted on toolpost 105 for releasibly engaging housing 88 and for coupling a surface-controlled hydraulic impact wrench to an end portion 82 of shaft 80. Assembly 160 may thus be used for adjusting a pair of locking assemblies 79 between the unlocked position shown in Figure 10 and the locked position shown in Figure 11.

As shown best in Figures 16 and 17, assembly 160 includes a pair of elongate hydraulic cylinder housings 162 which are each deployed directly above a respective lock housing 88 in the operative position. Cylinder housings 162 are connected by a pair of arcuate plates 164 which straddle toolpost 105 in parallel relation (Figure 17(a)). A pair of brackets 166 are mounted on toolpost 105 for capturing a central portion of arcuate plates 164. As best shown in Figure 17(b), the undersurface 168 of each plate 164 is notched as is the upper surface 170 of brackets 166. Brackets 166 are positioned on toolpost 105 such that the notches formed on plate lower surface 168 ordinarily ride clear of the mating notches formed on the bracket upper surface 170 to enable plates 164 to swing back and forth as described in further detail below. However, when

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toolpost 105 is lifted clear of clamping apparatus 10, the mating notches engage, thereby preventing swinging movement of lock activating assembly 160. This feature is important to ensure that assembly 160 does not become entangled with  
5 clamping apparatus 10.

As shown best in Figure 18, each housing 162 comprises a hollow structural section of tubing designed to receive a dual piston, dual rod hydraulic cylinder 172. As  
10 best shown in Figures 19(a) - 19(c), hydraulic cylinder 172 includes a mounting base 174 having four threaded mounting holes 176 and a central hydraulic oil inlet port 178 for actuating outward sliding movement of push rods 180. Two hydraulic oil inlet ports 182 are located at the extreme  
15 ends of cylinder 172 for actuating inward movement of push rods 180. The end portions of each push rod are threaded for receiving two lock nuts 184.

With reference to Figures 20(a) - 20(c), each  
20 cylinder housing 162 includes 4 apertures 186. Housing apertures 186 are aligned with corresponding apertures 176 machined in cylinder mounting base 174 and fasteners 187 are inserted therethrough for bolting cylinder 172 in place (Figure 18). Each housing 162 also includes a centrally  
25 located aperture 188 and two outwardly located apertures 190 which are alignable with cylinder mounting base apertures 176 and 182 respectively to allow for passage of the hydraulic oil connections 192 to the cylinder 172. Preferably, housing 162 has a slot 194 formed in its bottom end  
30 to facilitate placement of hydraulic cylinder 172 within housing 162. Alternatively, hydraulic cylinder 172 may be inserted into housing 162 through either of the open tube ends and then bolted into place as aforesaid.

35 Each cylinder housing 162 further includes 4 outwardly projecting lugs 196 located on the vertical side faces thereof (Figures 17, 20 and 21). Lugs 196 are pro-

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vided for engaging a pair of brackets 198 mounted on the top surface of each locking assembly housing 88 (Figures 15, 16 and 22). Lugs 196 and brackets 198 are provided for restraining longitudinal and lateral shifting of lock activating assembly 160 relative to locking assembly 79. Preferably the vertical faces of brackets 198 are relatively short to prevent binding when toolpost 105 is lifted clear of clamping apparatus 10.

As best shown in Figures 15, 16 and 18, lock activating assembly 160 further includes means for releasably engaging the end portions of lock housings 88. In particular, assembly 160 includes tubular components 200 and 202 which are slidably mounted on opposite ends of each housing 162 and are operatively connected to hydraulic cylinder 172 as described below. Two brackets 206 are secured to component 200 to hold a tool holder 208 in place. Component 200 further includes an internal plate 210 having an aperture formed therein which is captured by nuts 184 located on cylinder push rods 180 (Figure 18). A slot 214 is also formed in component 200 to prevent interference with the hydraulic lines 192 which extend through cylinder inlet apertures 182 and housing apertures 190 (Figure 24).

Tool holder 208 is designed to receive an end portion of lock assembly housing 88 as shown in Figures 15 and 16. A hydraulic impact wrench or other equivalent actuating tool (not shown) is fitted within tool holder 208 for engaging hexagonal end portion 82 of shaft 80, which rotates within housing 88 as described above.

Component 202, which is slidably mounted on the end of housing 162 opposite component 200, also includes an internal plate 210 for coupling to hydraulic cylinder 172 and a slot 214 (Figure 25). A plate 218 having a rectangular opening is secured to the outermost end of compo-

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nent 202. Plate 218 is provided for receiving an end portion of lock housing 88 opposite the end received by tool holder 208 (Figures 15 and 16).

5 As described in further detail below, operation of each hydraulic cylinder 172 may be remotely controlled by a surface operator to actuate sliding extension of components 200 and 202 between an operative position, shown in solid outline in Figure 23, wherein tool holder 208 and  
10 plate 218 engage respective ends of a housing 88, and an extended position, shown in dotted outline in Figure 23, wherein tool holder 208 and plate 218 are disengaged from housing 88. When tool holder 208 and plate 218 are in the disengaged position aforesaid, lock activating assembly 160  
15 may be lifted clear of clamping apparatus 10 when toolpost 105 is raised.

As indicated above, lock activating assembly 160 consists of two hydraulic cylinder housings 162 which are  
20 connected by arcuate plates 164 (Figures 16 and 17). Tool holders 208 connected to the respective cylinder housings 162 are preferably diagonally cross-mounted to maintain a weight balance when toolpost 105 and attached lock activating assembly 160 are lifted clear of clamping apparatus 10.  
25 With reference to Figure 16, relatively heavy tool holder 208 is shown on the left hand side of the drawing and relatively light plate 218 is shown on the right hand side. The reverse arrangement appears on the opposite side of clamping apparatus 10. With reference to Figure 17(a), it  
30 may be seen that the respective cylinder housings 162 are longitudinally offset. This offset is required to allow for the diagonally opposite mounting of respective tool holders 208 and plates 218 as described above.

35 In operation, clamping apparatus 10 is lowered on to a length of conduit 11 laid over an irregular sea bed. As shown schematically in Figure 1, clamping apparatus 10

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is typically lowered from a surface vessel such as a floating barge which can be maintained in a relatively stationary position over the installation site of interest. In the case of a barge this is usually done with anchors, cables and winches. Other surface vessels may be equipped with thrusters and a positioning system such as a satellite link or positioning beacons.

The surface vessel must be equipped with a crane capable of lowering clamping apparatus to the ocean bottom. In shallower waters a pre-deployment survey of a given site and the actual deployment may be accomplished by divers. In deeper waters underwater remote controlled equipment must be used, such as an underwater remote operated vehicle (ROV) fitted with cameras and having dual or multiple manipulator arms.

The installation site is initially surveyed by lowering a calibrated range pole in the vicinity of the pipe span to be supported. Depending upon the depth of deployment, the range pole is either lowered directly from the surface vessel, or it is mounted in a vertical orientation in one of the manipulator arms of the remote underwater vehicle. The other remote manipulator arm is typically fitted with a hydraulic impact wrench for actuation of the clamping apparatus clamp locking assembly. After the remote underwater vehicle has located the conduit, its exact location is determined. This is achieved by locating numbers pre-applied to conduit 11 as it was laid or from surveys undertaken after conduit laying. The installation site is monitored via a remote control camera as the range pole is placed next to conduit span 11. An elevation reading is then taken to determine the distance from conduit 11 to the ocean bottom. This information is then used to determine whether extensible leg members are long enough. In some instances supplementary leg extension pieces (not shown), connectable to inner legs

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112, are required. If the ocean bottom has a cross slope, clamping apparatus 10 may be fitted with one supplementary leg extension only.

5           While the installation site is surveyed as aforesaid by the ROV, clamping apparatus 10 is readied for use on the surface vessel. With reference to Figure 3, toolpost 105 is fitted onto tubular mounting post 44. Toolpost pin linkage 109 is fitted into aperture 46 formed  
10 in mounting post 44 to releasibly secure toolpost 105 to anchor frame 13. As previously described, toolpost 105 includes tool holders 108 for housing hydraulic impact wrenches or the like which may be fitted onto the hexagonal end portions 116 of leg screws 114 projecting from the  
15 upper ends of support leg housings 110. Toolpost 105 also functions as a mounting support for various other equipment such as cameras, lights and inclinometers.

Hydraulic power for the impact wrenches and other  
20 equipment is either supplied through hydraulic lines extending directly from the surface vessel, or from a submersible hydraulic power pack (not shown) connectible to toolpost 105. The hydraulic power pack is equipped with surface controlled valving and, when deployed, is preferably  
25 suspended in a cage by means of lifting slings approximately 25 feet above toolpost 105. Electric power and valving control wiring is provided to the hydraulic power pack from the surface by means of an umbilical cable spooled over the side of the surface vessel together with  
30 the ~~main crane support~~ cable. Conventional hydraulic lines are used to provide hydraulic flow from the power pack to the impact wrenches housed within tool holders 108 and 208.

Prior to lowering clamping apparatus 10 into the  
35 water, proper closing action of clamps 20 and seating of locking bolt 60 is preferably checked by lowering clamping apparatus 10 on to a test piece of conduit mounted on the

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surface vessel. Proper releasable mounting of toolpost 105 on mounting post 44 and operation of screw-activated extensible legs 40 should also be routinely tested at the surface prior to deployment. If not already pre-installed, 5 load transferring boots 128 are fitted to the ends of inner extensible legs 112.

Clamping apparatus 10 and connected toolpost 105, together with the hydraulic power pack (if required), are 10 then lowered over the side of the surface vessel. The depth of deployment of clamping apparatus 10 may be monitored by referring to distance markings on the crane cable. As clamping apparatus 10 approaches the ocean bottom, the cameras fitted on the remote underwater vehicle may be used 15 to track lights mounted on toolpost 105. Surface operators can thus monitor and control the descent of clamping apparatus 10 by suitably manipulating the surface vessel's crane as apparatus 10 is lowered toward the range pole.

20 With the aid of continued surface monitoring and control as aforesaid, clamping apparatus 10 is gradually lowered onto conduit span 11. Flared guide plates 37 extending from the lower portion of saddle 12 guide saddle 12 on to conduit 11, thus aligning the longitudinal axes 25 of saddle 12 and conduit 11.

As semi-circular saddle upper wall 16 is seated on the upper portion of conduit 11, clamps 20 are automatically swung into their closed positions engaging the 30 lower portion of conduit 11 (Figures 5-6 and 9-11). In particular, the second ends 30 of respective clamp halves 20(a) and 20(b) swing upwardly and outwardly as they contact conduit 11, thereby swinging clamp half first ends inwardly and downwardly toward one another through the 35 plane of saddle sidewalls 14. Swinging movement of clamp half 20(a) causes locking bolt 60 to assume a generally horizontal orientation underneath conduit 11 as shown in



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Figure 6. As described above, the exposed, free end of locking bolt 60 is pivotably carried into a slotted plate 33 as clamp halves 20(a), 20(b) close around conduit 11.

5           If conduit 11 is downwardly or upwardly inclined, anchor frame 13 will swing relative to saddle 12 about trunnions 48 as saddle 12 is seated on conduit 11. This ensures that support legs 40 will hang substantially vertically irrespective of the angle of inclination of  
10 conduit 11 and saddle 12 (Figure 12).

At this point, the ROV is used to conduct a visual inspection to ensure that clamps 20 are properly closed and that locking bolt 60 is correctly seated as  
15 aforesaid. Inclinometers (not shown) mounted on toolpost 105 are checked to ensure that anchor frame 13 is extending vertically. In the event of incorrect closure or seating, clamping apparatus 10 is simply lifted free of conduit 11 by the surface operated crane and reset. When  
20 clamping apparatus 10 is lifted clear of conduit 11 elastic cords 35 automatically urge clamps 20 to the open position. Stop bar 27 integrally connected to respective clamp hinge sleeves 24 prevents clamp halves 20(a), 20(b) from over-rotating past the preferred open position (Figure  
25 5).

Once it has been determined that clamping apparatus 10 is properly deployed, clamps 20 are locked in the closed position. In particular, the remote underwater  
30 vehicle is maneuvered to fit the hydraulic impact wrench pre-mounted in one of its manipulator arms onto the hexagonal coupling nut 70 secured to bolt 60. Another manipulator arm may be braced on frame cage 113 for added stability. Rotation of nut 70 and hence bolt 60 causes block nut  
35 66 to travel partially along the length of bolt 60, toward bolt housing 62, until nut 66 is seated securely against plate 33, thus locking clamp halves 20(a) and 20(b) to-

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gether (Figures 6 and 8). Other saddle clamps 20 are locked in similar fashion.

If clamping apparatus 10 is fitted with the alternative locking assembly illustrated in Figures 9-12, then saddle clamps 20 are locked by securing the ROV's hydraulic impact wrench to either one of the hexagonal end portions 82 of shaft 80. As previously described, rotation of shaft 80 causes nuts 90, 92 to travel outwardly, thus applying an inwardly directed clamping force to clamp halves 20(a) and 20(b) as shown in Figure 11.

In the further alternative embodiment of clamping apparatus 10 shown in Figure 14, which does not employ self-closing clamps 20, both closure and locking of clamp halves 20(a) and 20(b) is actuated by rotation of shaft 80 as described above.

In yet another alternative embodiment of the invention, lock activating assembly 160 mounted on toolpost 105 may be employed for locking clamps 20 in the closed position (Figures 15 and 16). Since assembly 160 may be remotely controlled by a surface operator, this avoids the need to use hydraulic wrenches pre-mounted on the R.O.V. to perform the screw-activating tasks. As indicated above, this is a significant advantage in the case of shallow spans where there is little clearance under saddle 20 to manoeuvre the R.O.V.

Lock activating assembly 160 is designed to enable remote control of the locking assemblies 79 shown in Figures 9 - 12 and 14 which are mounted above clamps 20. Lock activating assembly 160 is initially coupled to locking assemblies 79 on the surface vessel before clamping apparatus 10 and connected toolpost 105 are lowered into the water as described above. In particular, tool holders 208 of assembly 160 each receive one end of a

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respective lock housing 88 and plate 218 receives an opposite end of housing 88 (Figure 15). Each tool holder 208 houses a hydraulic impact wrench which is fitted on to a hexagonal end portion 82 of rod 80. As indicated above, 5 tool holders 208 are diagonally cross-mounted to maintain a weight balance on toolpost 105 (Figure 16).

Lock activating assembly 160 is designed to allow cylinder housings 162 and attached tool holders 208 to 10 swing relative to one another if clamping apparatus 10 is seated on an inclined pipe span 11. As shown best in Figure 16, respective cylinder housings 162 are connected by arcuate plates 164 which straddle toolpost 105. Plates 164 are freely captured within brackets 166 in the operative position (Figures 15 and 16). 15

After clamping apparatus 10 is properly seated on conduit 11 as described above, lock activating assembly 160 is actuated to adjust locking assemblies 79 between the 20 unlocked position shown in Figure 10 and the locked position shown in Figure 11. In particular, the hydraulic wrenches mounted within tool holders 208 are activated by the surface operator to cause rotation of rods 80 which in turn causes nuts 90, 92 to travel outwardly, thus applying 25 an inwardly directed clamping force to respective clamp halves 20(a) and 20(b) as shown in Figure 11.

Lock activating assembly 160 is then disengaged from lock housings 88 by the surface operator by actuating 30 outward extension of components 200 and 202 (Figure 23). The hydraulic wrenches housed within tool holders 208, and the dual piston, dual rod hydraulic cylinders 172 which actuate sliding movement of components 200 and 202, are preferably connected to hydraulic lines extending from the 35 surface vessel or a submersible hydraulic power pack which is connectible to toolpost 105 and is equipped with surface controlled valving.

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In the event that one of the hydraulic wrenches or other actuating mechanism mounted within tool holder 208 fails, it is possible to activate lock assembly 79 by  
5 securing a hydraulic wrench mounted on a manipulator arm of the R.O.V. to the opposite end of rod 80 through plate 218. Support legs 40 may then be extended as described above by activating the hydraulic wrenches housed within tool holders 108. Toolpost 105 may then be disengaged from  
10 clamping apparatus 10 and raised to the surface vessel. Alternatively, in the event of a serious tool failing, the toolpost 105 may be immediately disengaged from apparatus 10 and all of the screw-activating tasks may be performed by the R.O.V.

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As should be apparent to someone skilled in the art, in shallower water, locking of clamps 20 may be accomplished by divers carrying hydraulic impact wrenches rather than via a remote underwater vehicle or remote controlled  
20 lock activating assembly 160.

As clamping apparatus 10 is seated on conduit 11 the weight of apparatus 10 is transferred from the ship board crane to conduit 11. This added load on conduit 11  
25 causes a deflection in the span. Accordingly, after clamps 20 are securely locked in the closed position as described above, a reading is taken on the previously positioned range pole to determine how much conduit 11 must be raised to return it to its original position. This distance is  
30 added to a pre-calculated distance to reflect the loading requirements of the frame support legs 40. One of frame support legs 40 is then extended as described above. In particular, power is supplied to one of the hydraulic wrenches mounted within tool holders 108 (mounted on tool-  
35 post 105) to rotate the associated leg screw 114, telescopically extending inner leg 112 relative to outer housing 110 (Figure 13). As boot 128 contacts the seabed,

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anchor frame 13 will begin to support part of the load of conduit 11, causing conduit 11 to deflect upwardly and laterally.

Once approximately half the required height correction has been achieved by extending one of the inner legs 112 as aforesaid, extension of that inner leg 112 stops. The inner leg 112 of the other support leg 40 is then extended in similar fashion thus further raising and deflecting conduit 11 in a balanced manner until conduit 11 is returned to its original position. Final adjustments are made to each of the respective support legs 40 until the correct distance reading is indicated on the range pole.

After a final visual check is made via the ROV, toolpost linkage pin 109 is released, either by actuation of the hydraulic ram mounted on toolpost 105, manually by an underwater diver, or via an ROV manipulator arm. Toolpost 105 and attached tool holders 108 are then raised clear of clamping apparatus 10 by the surface operated crane.

If a lock activating assembly 160 is mounted on toolpost 105, then the notched upper surface of brackets 166 will engage the mating notches formed on the undersurface of arcuate plates 164 as toolpost 105 is raised by the surface operated crane. This ensures that assembly 160 does not become entangled with clamping apparatus 10 as toolpost 105 is raised. This feature is particularly important if saddle 12 is deployed on an inclined conduit 11 (Figure 12) which would result in a weight imbalance on respective ends of assembly 160 (Figures 16 and 17).

After toolpost 105 and the hydraulic power pack are raised to the surface vessel, they may be quickly connected to another clamping apparatus 10. The deployment procedure detailed above is then repeated.

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Following the procedure described above, installation of clamping apparatus 10 may be accomplished much more quickly than conventional grout bag supports. For example, in one representative session 4 units were installed in less than 24 hours at water depths approaching 1400 feet.

As will be apparent to those skilled in the art in the light of the foregoing disclosure, many alterations and modifications are possible in the practice of this invention without departing from the spirit or scope thereof. Accordingly, the scope of the invention is to be construed in accordance with the substance defined by the following claims.

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## WHAT IS CLAIMED IS:

1. A self-closing clamping apparatus (10) for engaging and supporting an elongate body (11), such as a length of conduit, said apparatus comprising a saddle (12) shaped to conform to an exterior surface of said elongate body (11), said saddle (12) having opposed side portions (14) defining a compartment (15) therebetween for receiving said elongate body (11), characterized in that said clamping apparatus further comprises clamping means hingedly connected to said saddle (12) and swingable between an open position wherein a portion of said clamping means extends within said compartment (15) and a closed position clampingly engaging a portion of said elongate body exterior surface, wherein placement of said elongate body (11) into said compartment (15) as said saddle (12) is seated thereon forces said clamping means portion outwardly, thereby causing said clamping means to swing from said open position to said closed position.
2. A clamping apparatus (10) as defined in claim 1, wherein said clamping means comprises a pair of clamps (20(a),20(b)), each clamp (20(a),20(b)) being hingedly connected to one of said saddle side portions (14).
3. A clamping apparatus (10) as defined in claim 2, wherein said saddle side portions (14) further comprise slots (18) to permit swinging movement of said clamps (20(a),20(b)) between said open and closed positions through said saddle side portions (14).
4. A clamping apparatus (10) as defined in claim 3, wherein each of said clamps (20(a),20(b)) comprise a clamping surface (31) shaped to conform to said elongate body exterior surface, said clamping surface (31) having a first end (28) extending outwardly of said saddle side portions (14) in said open position and a second end (30)

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comprising said clamping means portion extending within said compartment (15) in said open position.

5. A clamping apparatus (10) as defined in claim 4,  
5 wherein placement of said elongate body (11) into said compartment (15) as said saddle (12) is seated on said elongate body (11) forces said clamping surface second ends (30) outwardly away from one another, thereby swinging said clamping surface first ends (28) inwardly toward one  
10 another.

6. A clamping apparatus (10) as defined in claim 1,  
further comprising biasing means for urging said clamping means towards said open position.

15

7. A clamping apparatus as defined in claim 4,  
further comprising biasing means for urging said clamping means toward said open position, said biasing means comprising a tensile member (35) connectible between said  
20 clamping surface first ends (28).

8. A clamping apparatus (10) as defined in claim 1,  
further comprising locking means for releasibly locking said clamping means in said closed position.

25

9. A clamping apparatus (10) as defined in claim 4,  
further comprising locking means for releasibly locking said clamping surface first ends (28) together.

30 10. A clamping apparatus (10) as defined in claim 9,  
wherein said locking means comprises a bolt (60) rotatably coupled to one of said clamps (20(a), 20(b)) and swingable therewith.

35 11. A clamping apparatus (10) as defined in claim 10,  
wherein said locking means further comprises:



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a) a housing (62) integrally connected to one of said clamps (20(a)) for rotatably receiving an end portion of said bolt (60), wherein said housing (62) is adapted for restricting axial travel of said bolt (60); and

b) a nut (70) threadedly connected to said bolt (60),

wherein rotation of said bolt (60) when said clamps (20(a), 20(b)) are in said closed position causes axial travel of said nut (70) between an unlocked position remote from said housing (62) and a locked position seated securely against the other of said clamps (20(b)) proximate said housing (62).

12. A clamping apparatus (10) as defined in claim 11, wherein said other clamp (20(b)) comprises a plate (33) extending from its first end (28) having a slot formed therein for receiving a central portion of said bolt (60) when said clamps (20(a), 20(b)) are in said closed position, wherein said nut (70) is seated against said plate (33) in said locked position.

13. A clamping apparatus (10) as defined in claim 11, wherein said other clamp (20(b)) further comprises means for restricting rotation of said nut (70) when said clamps (20(a), 20(b)) are in said closed position.

14. A clamping apparatus (10) as defined in claim 8, wherein said locking means comprises:

- a) a housing (88) mounted on said saddle (12);
- b) a bolt (80) rotatable within said housing (88); and
- c) linkage means for operatively connecting said bolt (80) to said clamping means.

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15. A clamping apparatus (10) as defined in claim 2, further comprising locking means for releasibly locking said pair of clamps (20(a),20(b)) in said closed position, said locking means comprising:

5

- a) a housing (88) mounted on said saddle (12);
- b) a bolt (80) rotatable within said housing (88); and
- c) linkage means (96) for operatively connecting said bolt (80) to each of said pair of clamps (20(a),-

10

20(b)).

16. A clamping apparatus (10) as defined in claim 15, wherein said bolt (80) has a central portion and left hand threaded (84) and right hand threaded portions (86), said linkage means comprising:

15

- a) a pair of nuts (90,92), wherein each nut (90,92) is threadedly connected to one of said left hand threaded (84) or right hand threaded (86) portions;
- c) a pair of hinge pins (104), wherein each hinge pin (104) is connected to one of said clamps (20(a),20(b)) and is movable therewith;
- d) linkage arms (96) for operatively connecting said nuts (90,92) to said hinge pins (104), each linkage arm (96) having a first end (98) coupled to one of said nuts (90,92) and a second end (99) coupled to one of said hinge pins (104),

20

25

wherein rotation of said bolt (80) within said housing (88) causes outward travel of said nuts (90,92) toward a locked position remote from said central portion and counter-rotation of said bolt (80) causes inward travel of said nuts (90,92) toward an unlocked position proximate said central portion.

35

17. A clamping apparatus (10) as defined in claim 16, wherein said linkage arm second ends (99) further comprise

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slots (101) to enable sliding movement of said hinge pins (104) relative to said linkage arms (96) when said clamps (20(a), 20(b)) swing between said open and closed positions.

5

18. A clamping apparatus (10) as defined in claim 2, wherein said elongate body (11) comprises a length of cylindrical conduit elevated above a support surface, wherein said saddle (12) comprises a semi-cylindrical upper surface (16) extending between said saddle side portions (14) for overlying an upper portion of said conduit, and wherein each of said clamps (20(a), 20(b)) comprise an arcuate clamping surface (31) for engaging at least a lower portion of said conduit (11) in said closed position.

15

19. A clamping apparatus (10) as defined in claim 18, further comprising anchor means coupled to said saddle (12) for anchoring said clamping apparatus (10) on said support surface.

20

20. A clamping apparatus (10) as defined in claim 19, wherein said anchor means further comprises a pair of support legs (40), each support leg (40) being pivotally connected to one of said saddle side portions (14), wherein said support legs (40) are adapted for extending between said elevated conduit and said support surface.

21. A clamping apparatus (10) as defined in claim 20, wherein the centre of gravity of said support legs (40) is below the point of pivotal connection of said support legs (40) to said saddle (12), whereby said support legs (40) extend in a substantially vertical plane irrespective of the angle of inclination of said saddle (12) relative to said support surface.

35

22. A clamping apparatus (10) as defined in claim 20, wherein said support legs (40) are angled outwardly rela-

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tive to said saddle (12) to provide lateral support for said elevated conduit.

23. A clamping apparatus (10) as defined in claim 20,  
5 wherein said support legs (40) comprise telescopic shafts (112) adjustable between retracted and deployed positions.

24. A clamping apparatus (10) as defined in 23,  
wherein said support legs (40) further comprise elongated  
10 screws (114) rotatable within said respective shafts (112) for adjusting said shafts (112) between said retracted and deployed positions.

25. A clamping apparatus (10) as defined in claim 24,  
15 further comprising means on said clamping apparatus (10) for releasibly coupling a hydraulic wrench to said screws (114).

26. A clamping apparatus (10) as defined in claim 20,  
20 wherein said support legs (40) further comprise flared boots (128) mounted at a lower end thereof for securely engaging said support surface and for spreading the load of said conduit on said support surface.

25 27. A clamping apparatus (10) as defined in claim 20, wherein upper portions of said support legs (40) are integrally joined by a transverse crosspiece (42), said support legs (40) and said crosspiece (42) together defining a swingable yoke straddling said saddle (12).

30

28. A clamping apparatus (10) as defined in claim 27, wherein said crosspiece (42) further comprises an upright mounting post (44) to facilitate placement of said saddle (12) on said conduit (11).

35

29. A clamping apparatus (10) as defined in claim 1, further comprising guide means integrally connected to said

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saddle side portions (14) for guiding said conduit (11) into said compartment (15).

30. A clamping apparatus (10) for engaging an elongate body (11) and for supporting said elongate body (11) on a support surface, said apparatus comprising a saddle (12) shaped to conform to an exterior surface of said elongate body (11), said saddle having opposed side portions (14) defining a compartment (15) therebetween for receiving said elongate body (11), characterized in that said apparatus further comprises:

15 a) clamping means mounted on said saddle (12) for clampingly engaging said elongate body (11); and

b) anchor means coupled to said saddle (12) for extending from said saddle (12) to said support surface, said anchor means being pivotable about a pivot axis intersecting said saddle side portions (14).

31. A clamping apparatus (10) as defined in claim 30, wherein said clamping means is hingedly connected to said saddle (12) and is swingable relative to said saddle (12) between an open position extending outwardly of said saddle (12) and a closed position clamping said elongate body exterior surface.

32. A clamping apparatus (10) as defined in claim 31, further comprising locking means for releasibly locking said clamping means in said closed position.

33. A clamping apparatus (10) as defined in claim 32, wherein actuation of said locking means causes swinging movement of said clamping means.

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34. A clamping apparatus (10) as defined in claim 33, wherein said locking means comprises:

- a) a housing (88) mounted on said saddle (12);
- 5 b) a bolt (80) rotatable within said housing (88); and
- c) linkage means for operatively connecting said bolt (80) to said clamping means.

35. A clamping apparatus as defined in claim 31, wherein said elongate body (11) comprises a length of cylindrical conduit elevated above said support surface, wherein said saddle (12) comprises opposed side portions (14) and a semi-cylindrical upper surface (16) extending between said saddle side portions (14) for overlying an upper portion of said conduit, said clamping means comprising a pair of clamps (20(a),20(b)), each clamp (20(a),- 20(b)) being hingedly connected to one of said saddle side portions (14).

20 36. A clamping apparatus (10) as defined in claim 35, further comprising locking means for releasibly locking said pair of clamps (20(a),20(b)) in said closed position, said locking means comprising:

- 25 a) a housing (88) mounted on said saddle (12);
- b) a bolt (80) rotatable within said housing (88); and
- c) linkage means for operatively connecting said bolt (80) to each of said pair of clamps (20(a),20(b)).

30 37. A clamping apparatus (10) as defined in claim 36, wherein said bolt (80) has a central portion and left hand threaded (84) and right hand threaded portions (86), said linkage means comprising:

- 35 a) a pair of nuts (90,92), wherein each nut (90,92) is threadedly connected to one of said left hand threaded (84) or right hand threaded (86) portions;

- 45 -

- c) a pair of hinge pins (104), wherein each hinge pin (104) is connected to one of said clamps (20) and is movable therewith;
- d) linkage arms (96) for operatively connecting said nuts (90,92) to said hinge pins (104), each linkage arm (96) having a first end (98) coupled to one of said nuts (90,92) and a second end (99) coupled to one of said hinge pins (104),
- 5
- 10 wherein rotation of said bolt (80) within said housing (88) causes outward travel of said nuts (90,92) toward a locked position remote from said central portion and counter-rotation of said bolt (80) causes inward travel of said nuts (90,92) toward an unlocked position proximate said
- 15 central portion.

38. A clamping apparatus (10) as defined in claim 37, wherein said linkage arm second ends (99) further comprise slots to enable sliding movement of said hinge pins (104) relative to said linkage arms (96) when said clamps (20) swing between said open and closed positions.

20

39. A clamping apparatus (10) for engaging an elongate body (11) and for connecting said elongate body (11) to a support surface, said apparatus comprising a frame (12) having opposed side portions (14) for receiving said elongate body (11) therebetween, characterized in that said apparatus further comprises:

25

(a) clamping means hingedly coupled to said frame (12) for clampingly engaging said elongate body (11), said clamping means being swingable about a hinge axis between open and closed positions; and

30

(b) anchor means coupled to said frame (12) for extending from said frame (12) to said support

35

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surface, said anchor means being pivotable about  
a pivot axis,

wherein said hinge axis and said pivot axis extend in non-  
5 parallel planes.

40. A clamping apparatus (10) as defined in claim  
39, wherein said pivot axis intersects said frame side  
portions (14).

10

41. A clamping apparatus (10) as defined in claim  
39, wherein said pivot axis extends in a plane perpen-  
dicular to a vertical plane passing through said hinge  
axis.

15

42. A clamping apparatus (10) as defined in claim  
39, wherein said elongate body (11) comprises a length of  
conduit elevated above said support surface, and wherein  
said anchor means comprises extensible legs (40) pivotally  
20 coupled to said frame side portions (14) for extending from  
said frame (12) to said support surface.

43. A clamping apparatus (10) as defined in claim  
42, wherein the centre of gravity of said anchor means is  
25 below said frame (12), whereby said extensible legs (40)  
hang in a vertical orientation irrespective of the angle of  
inclination of said frame (12).

44. A clamping apparatus (10) as defined in claim  
30 42, wherein said clamping means comprises a ~~clamp~~ portion  
extending at least partially between said frame side  
portions (14) in said open position, wherein said clamp  
portion contacts said conduit when said clamping apparatus  
(10) is seated thereon, thereby causing said clamping means  
35 to swing about said hinge axis to said closed position  
wherein said clamping means engages at least a lower  
portion of said conduit.



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45. A clamping apparatus (10) as defined in claim 44, wherein said clamping means comprises a pair of clamps (20(a),20(b)), each clamp (20(a),20(b)) being hingedly  
5 connected to one of said frame side portions (14) and comprising a clamping surface (31) shaped to conform to the exterior surface of said conduit.

46. A clamping apparatus (10) as defined in claim  
10 44, wherein said frame is shaped to conform to said conduit exterior surface.

47. A clamping apparatus (10) as defined in claim 44, further comprising biasing means for urging said  
15 clamping means toward said open position.

48. A clamping apparatus (10) as defined in claim 44, further comprising locking means for releasibly locking  
20 said clamping means in said closed position.

49. A clamping apparatus (10) as defined in claim 46 wherein said locking means is mounted on an upper  
portion of said frame (12).

25 50. A clamping apparatus as defined in claim 28, further comprising locking means for releasibly locking said pair of clamps (20(a),20(b)) in said closed position, said locking means comprising:

30 a) a housing (88) mounted on said saddle (12);  
b) a bolt (80) rotatable within said housing (88); and  
c) linkage means for operatively connecting said bolt (80) to each of said pair of clamps (20).

35 51. A clamping apparatus (10) as defined in claim 50, further comprising a toolpost (105) removably mountable on said upright mounting post (44), said toolpost (105) com-

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prising a suspension cable for retrieving said toolpost (105) from an underwater location, and remotely controllable actuating means mounted on said toolpost (105) for releasibly engaging and actuating said locking means.

5

52. A clamping apparatus (10) as defined in claim 51, wherein said actuating means further comprises a displaceable tool holder (208) for engaging an end of said housing (88) and means mounted within said tool holder (208) for  
10 actuating rotation of said bolt (80).

53. A clamping apparatus (10) as defined in claim 52, wherein said actuating means further comprises means mounted on said toolpost (105) for actuating extension of  
15 said support legs (40).

54. A toolpost assembly (160) for use in conjunction with a mechanical clamping apparatus (10) of the type comprising a frame (12), clamping means coupled to said  
20 frame (12) for engaging an elongate body (11), said clamping means being adjustable between an open position and a closed position clamping said elongate body (11), locking means for releasibly locking said clamping means in said closed position, and extensible anchor means pivotably  
25 coupled to said frame (12), characterized in that said toolpost assembly (160) comprises:

(a) a toolpost (105) for releasibly coupling said assembly (160) to said clamping apparatus (10), said toolpost (105) comprising a suspension  
30 cable for retrieving said toolpost (105) from an underwater location; and

(b) remotely controllable actuating means mounted  
35 on said toolpost (105) for releasibly engaging and actuating said locking means.

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55. A toolpost assembly (160) as defined in claim 54, wherein said locking means comprises a housing (88) mounted on said frame (12) above said clamping means; a bolt (80) rotatable within said housing (88); and linkage means for  
5 operatively connecting said bolt (80) to said clamping means, and wherein said actuating means further comprises a displaceable tool holder (208) for engaging an end of said housing (88) and means mounted within said tool holder (208) for actuating rotation of said bolt (80).

10

56. A toolpost assembly (160) as defined in claim 55, wherein said actuating means further comprises a remotely controllable hydraulic cylinder (172) for slidably adjusting said tool holder (208) along an axis coaxial with said  
15 bolt (80) between a retracted position engaging said housing (88) and an extended position clear of said housing (88).

57. A toolpost assembly (160) as defined in claim 56,  
20 wherein said clamping means comprises a first pair of clamps (20(a), 20(b)) and a second pair of clamps (20(a), 20(b)), and said locking means comprises a first bolt housing (88) mounted on said frame (12) above said first pair of clamps (20(a), 20(b)) and a second bolt housing (88)  
25 mounted on said frame above said second pair of clamps (20(a), 20(b)), and wherein said actuating means further comprises first and second hydraulic cylinder housings (162) each for receiving one of said hydraulic cylinders (172) and for resting on one of said bolt housings (88),  
30 and arcuate brackets (164) extending between said first and second cylinder housings (162) on opposite sides of said toolpost (105).

58. A toolpost assembly (160) as defined in claim 57,  
35 wherein the undersurface (168) of each of said arcuate brackets (164) is notched for securely engaging said

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toolpost (105) when said toolpost (105) is lifted clear of said clamping apparatus (10).

59. A toolpost assembly (160) as defined in claim 54,  
5 wherein said actuating means further comprises means mounted on said toolpost (105) for actuating extension of said anchor means.

60. A toolpost assembly (160) as defined in claim 59,  
10 wherein said anchor means comprises a pair of telescopically extensible support legs (40), and wherein said actuating means comprises hydraulic wrenches releasibly connectable to an upper portion of said support legs (40) for actuating extension thereof.

15  
61. A toolpost assembly (160) as defined in claim 60, wherein upper portions of said support legs (40) are integrally joined by a transverse crosspiece (42), said support legs (40) and said crosspiece (42) defining a  
20 swingable yoke straddling said frame, and wherein said toolpost (105) is releasibly connectable to an upper portion of said crosspiece (42).

62. A method of supporting a portion of an elongate  
25 body (11) in an elevated position above a support surface, said method comprising the steps of:

(a) providing a clamping apparatus (10) of the type comprising a frame (12), clamping means hingedly  
30 coupled to said frame (12) for engaging said elongate body portion (11), said clamping means being adjustable between an open position and a closed position clamping said elongate body (11), locking means for releasibly locking said clamping means in said closed position, and a pair of  
35 extensible support legs (40) pivotably coupled to

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said frame (12) for extending from said frame (12) to said support surface;

5 (b) determining the preferred elevation of said elongate body portion (11) above said support surface;

10 (c) suspending said clamping apparatus (10) above said elongate body portion (11) to be supported;

(d) lowering said clamping apparatus (10) onto said elongate body portion (11) until said clamping means securely engages said elongate body portion (11);

15 (e) actuating said locking means to securely lock said clamping means in said closed position;

20 (f) extending one of said support legs (40) until said leg (40) contacts said support surface, thereby resulting in upward deflection of said elongate body portion (11);

25 (g) extending the other of said support legs (40) until said other leg (40) contacts said support surface, thereby resulting in further upward deflection of said elongate body portion (11);  
and

30 (i) repeatedly extending said support legs (40) until said elongate body portion (11) is raised to its preferred elevation as determined in step (a).

63. The method as defined in claim 62, further  
35 comprising the steps of:

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- (a) measuring the initial elevation of said elongate body portion (11) above said support surface before said clamping apparatus (10) is lowered thereon;
- 5 (b) measuring the downward deflection of said elongate body portion (11) below said initial elevation after said clamping apparatus (10) has been lowered thereon; and
- 10 (c) calculating the preferred distance for raising said elongate body portion (11) based on the measurements made in steps (a) and (b) and the predetermined loading requirements of said extensible support legs (40).

15

64. A method of supporting a portion of an elongate body (11) in an elevated position above a support surface, said method comprising the steps of:

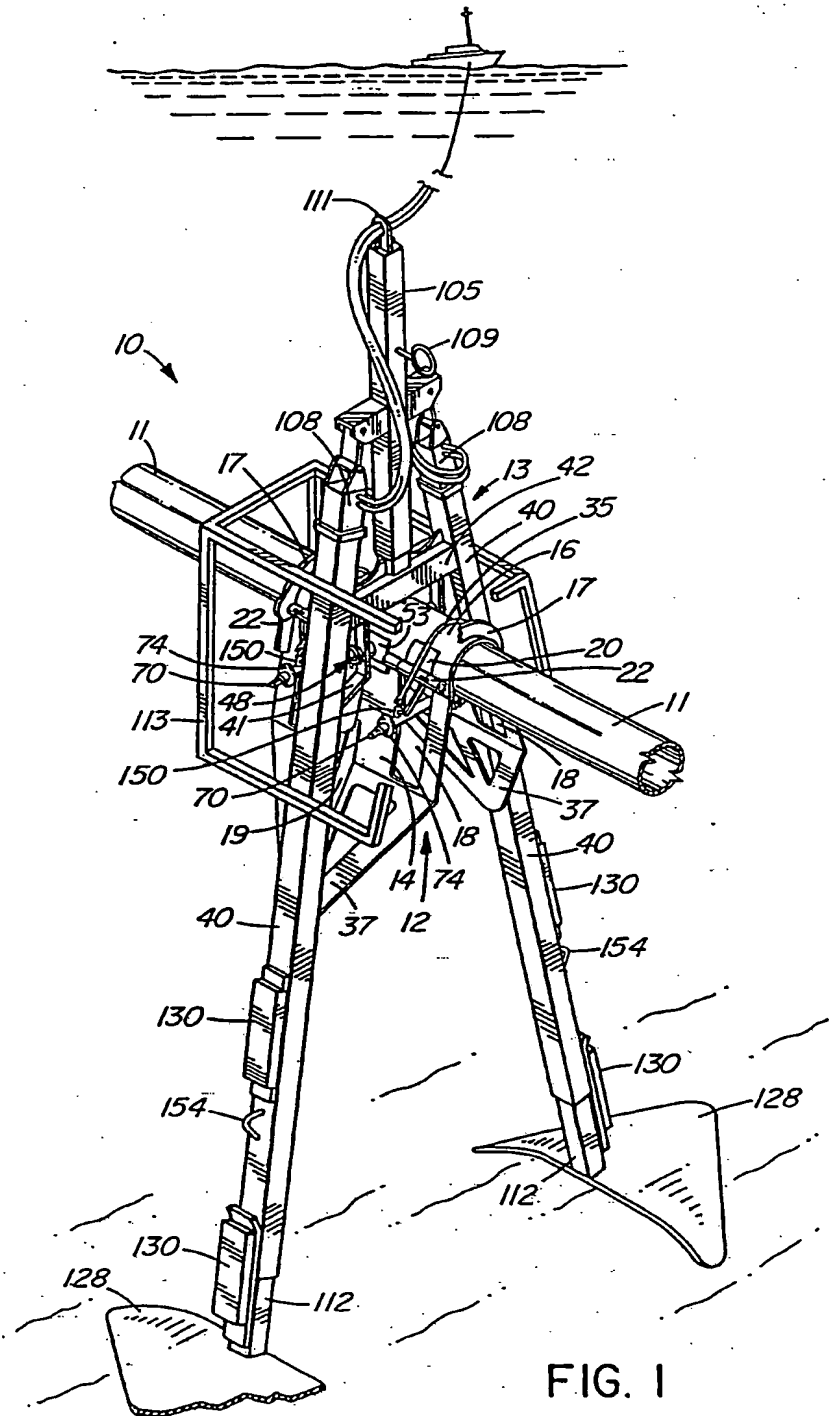
- 20 (a) providing a clamping apparatus (10) of the type comprising a frame (12), clamping means coupled to said frame (12) for engaging said elongate body portion (11), said clamping means being adjustable between an open position and a closed position clasping said elongate body (11),
- 25 locking means for releasibly locking said clamping means in said closed position, and extensible anchor means pivotably coupled to said frame (12) for extending from said frame (12) to said support surface;
- 30
- (b) providing a toolpost assembly (160) comprising a toolpost (105) for releasibly coupling said assembly (160) to said clamping apparatus (10),
- 35 and remotely controllable actuating means mounted on said toolpost (105) for releasibly engaging

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and actuating said locking means and for engaging and actuating extension of said anchor means;

- 5 (c) removably coupling said toolpost assembly (160) to said clamping apparatus (10) and suspending said assembly (160) and clamping apparatus (10) above said elongate body portion (11) to be supported;
- 10 (d) lowering said toolpost assembly (160) and clamping apparatus (10) onto said elongate body portion (11);
- 15 (e) actuating said toolpost actuating means by remote control to securely lock said clamping means in said closed position;
- 20 (f) actuating said toolpost actuating means by remote control to extend said anchor means to said support surface; and
- 25 (g) decoupling said toolpost assembly (160) from said clamping apparatus (10) and raising said toolpost assembly (160) clear of said clamping apparatus (10).

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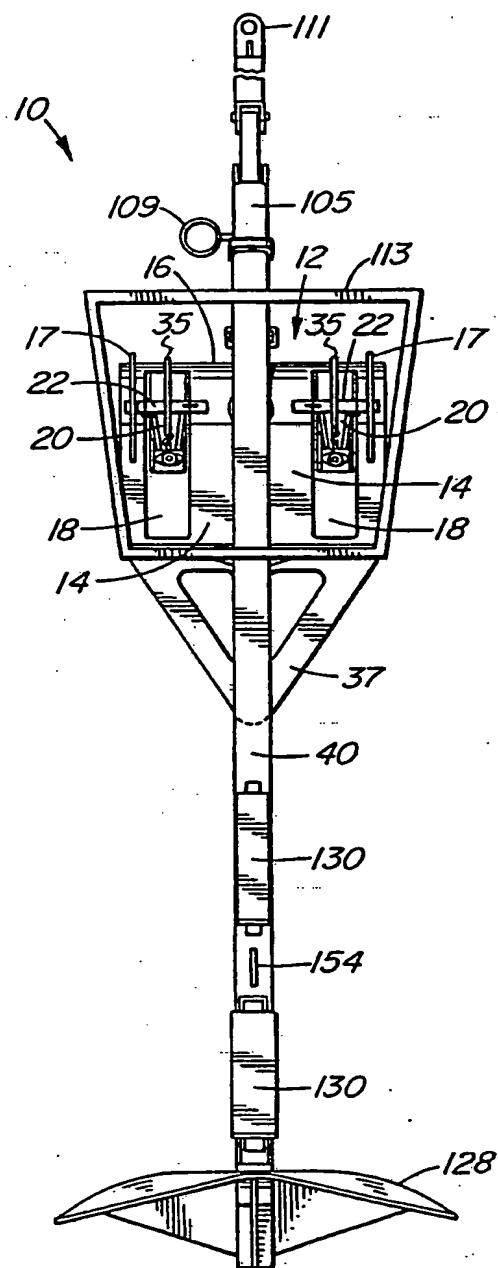
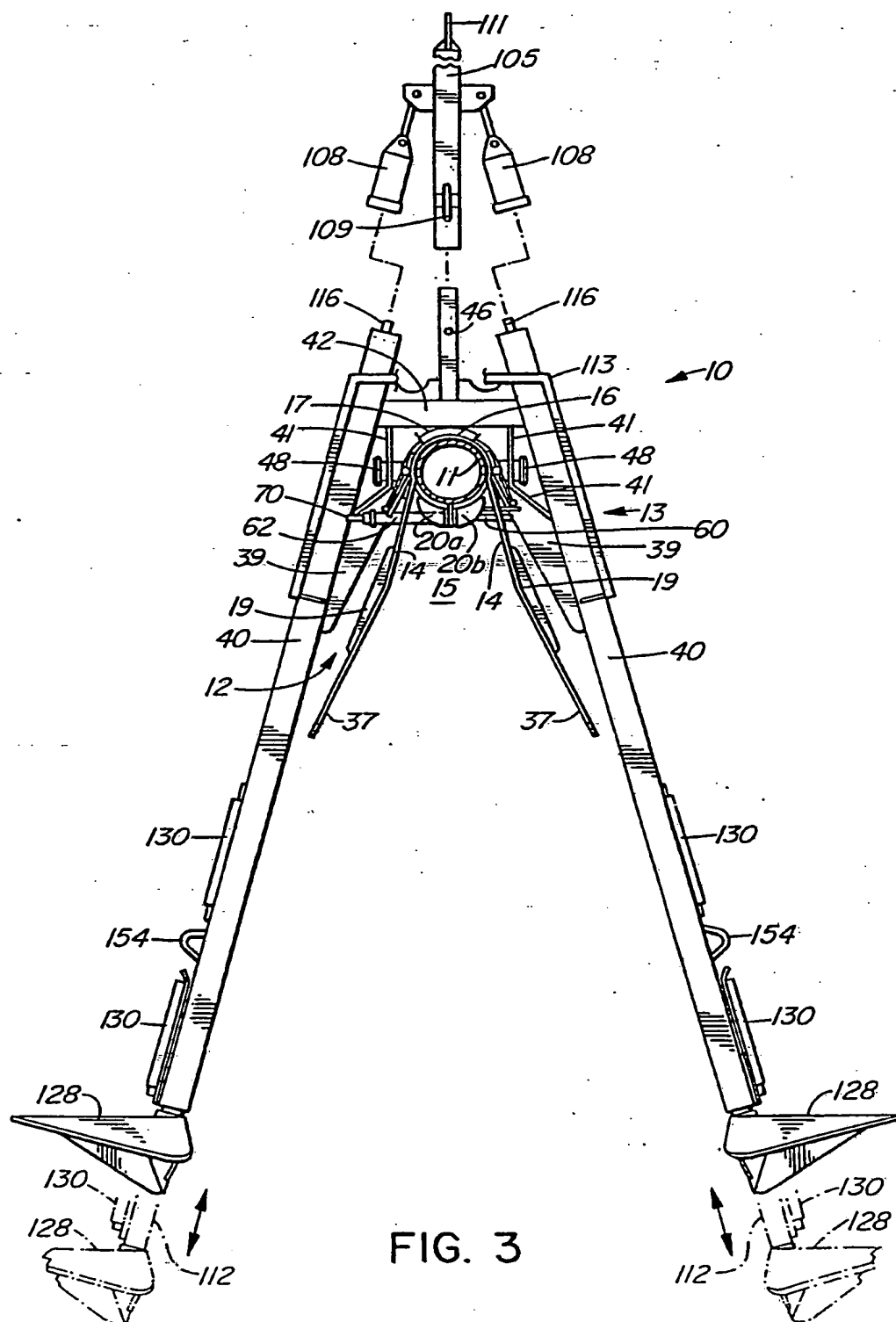


FIG. 2

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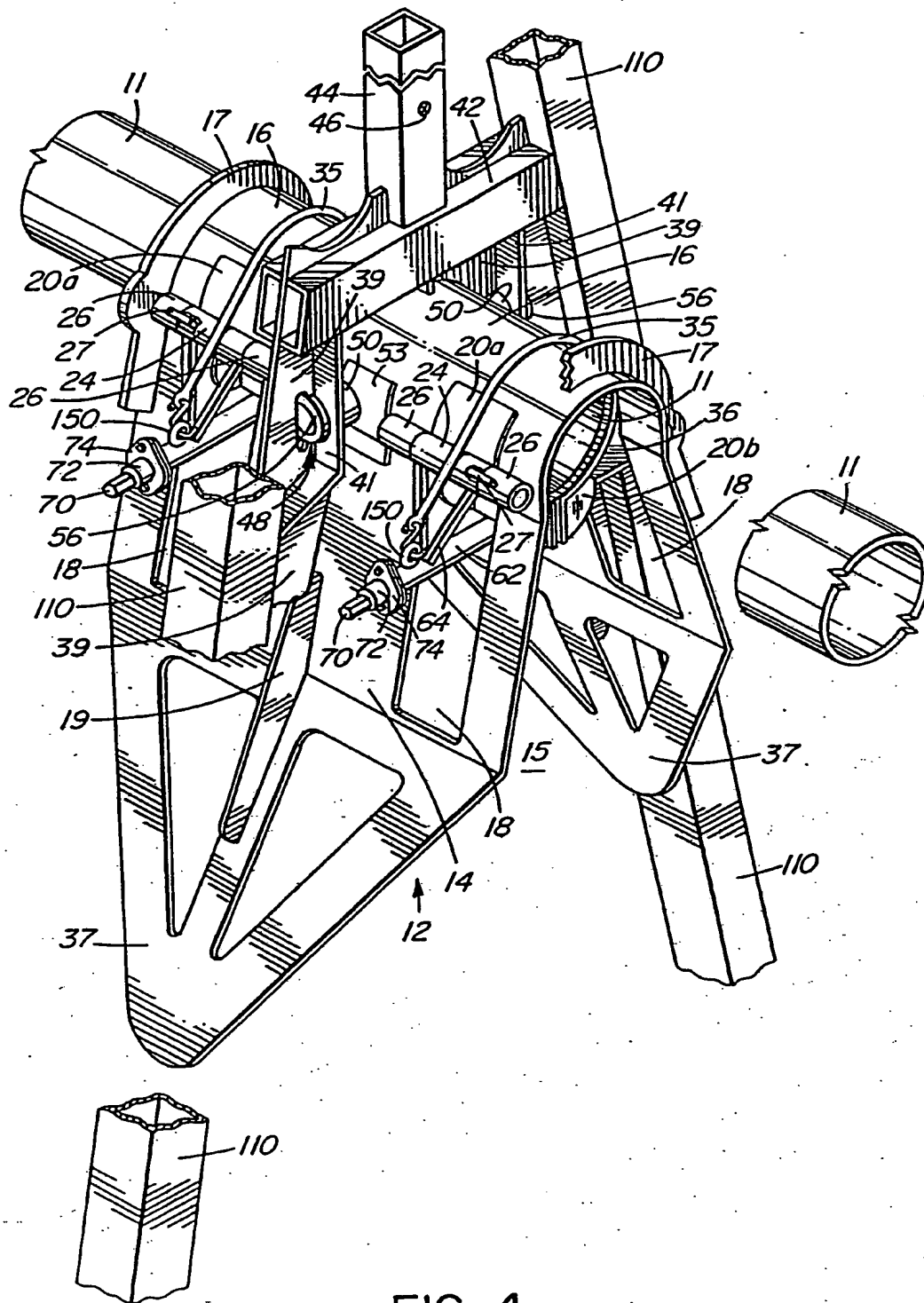


FIG. 4

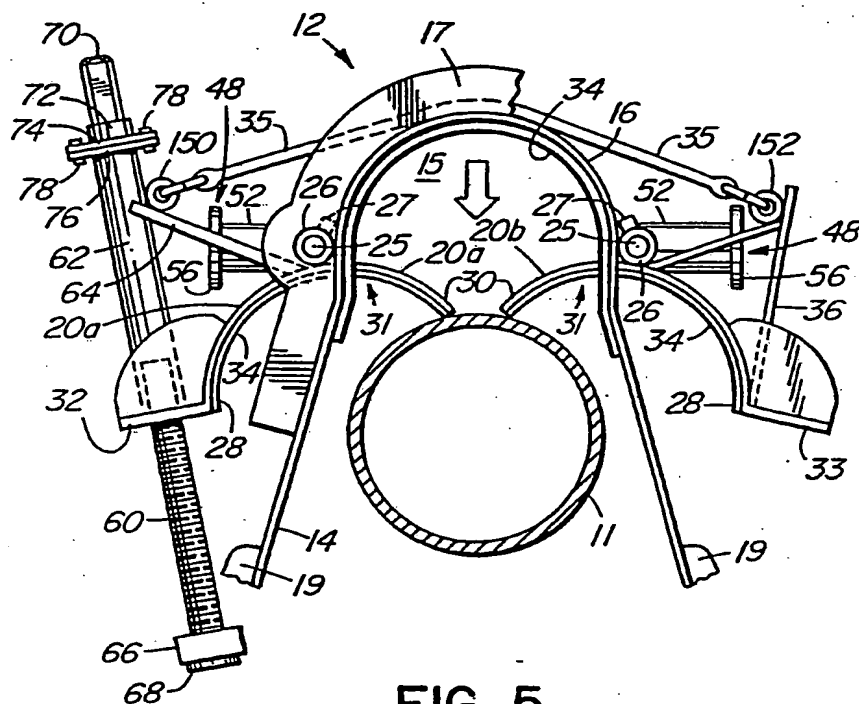


FIG. 5

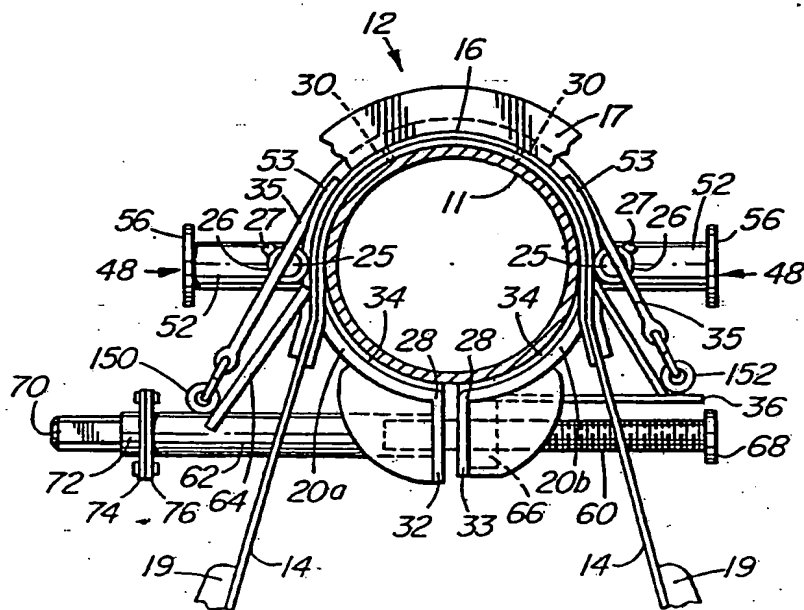


FIG. 6.

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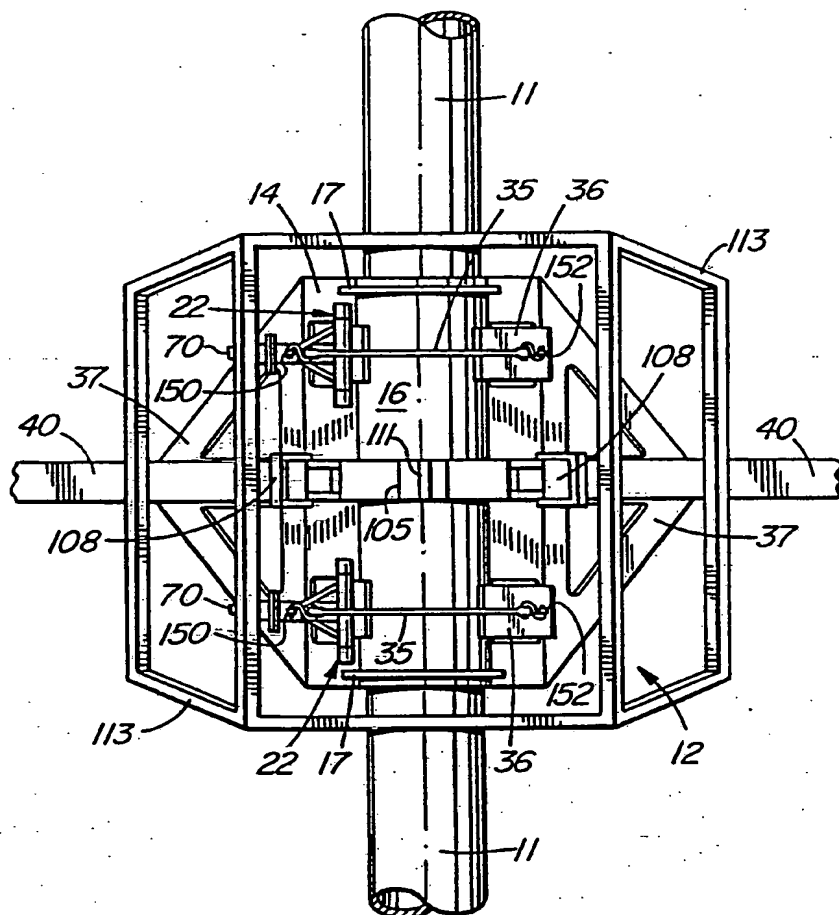


FIG. 7

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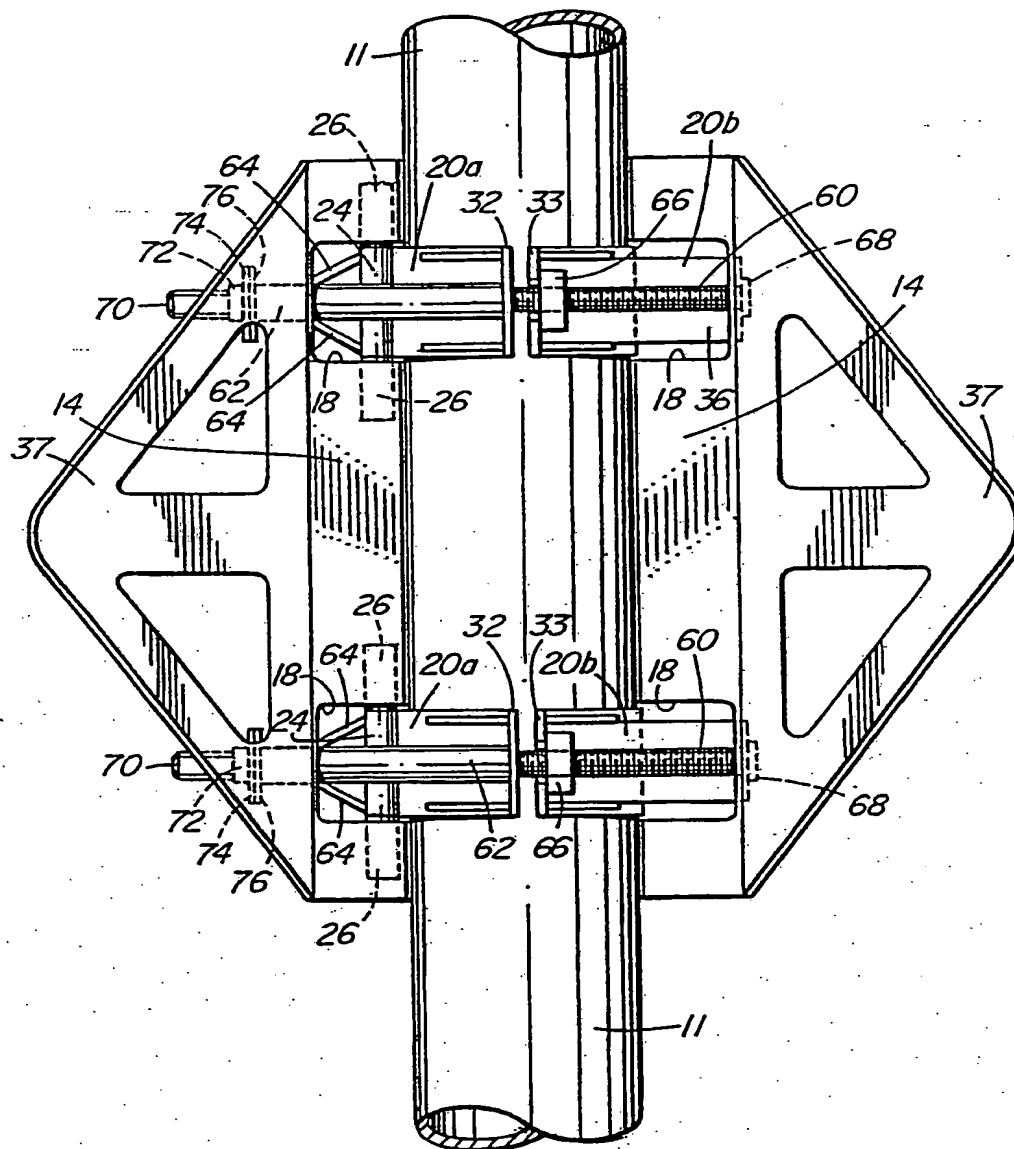


FIG. 8

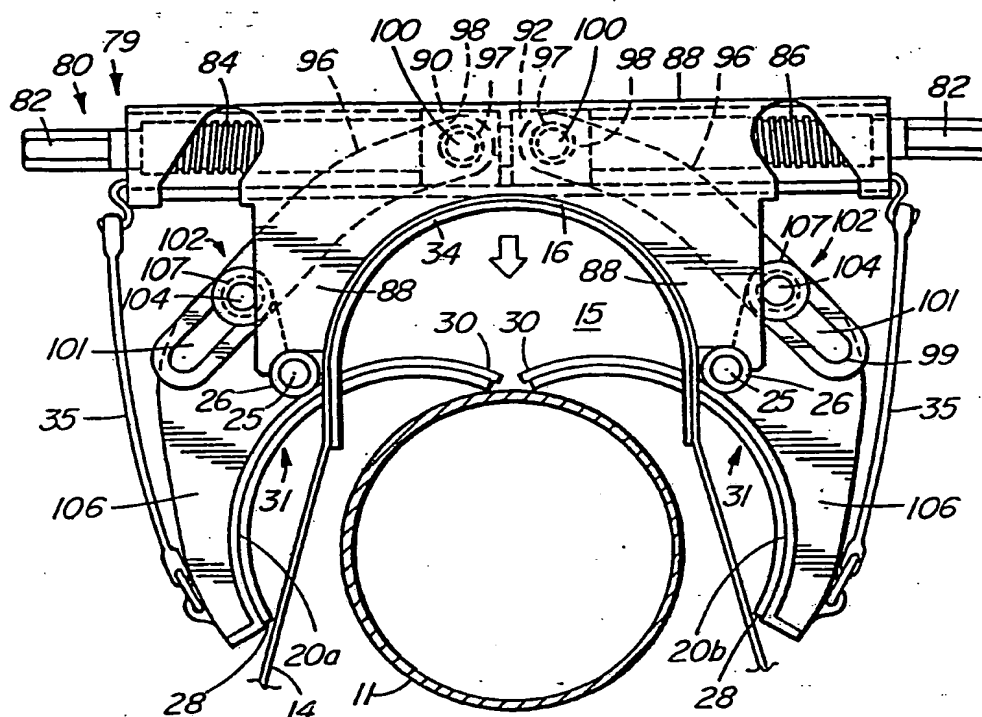


FIG. 9

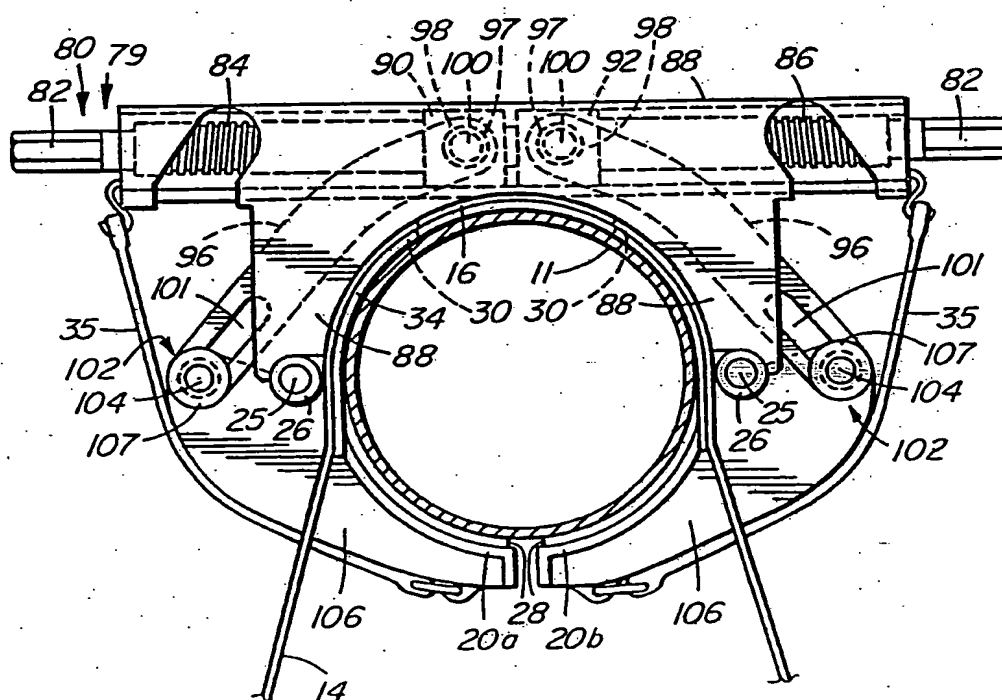


FIG. 10





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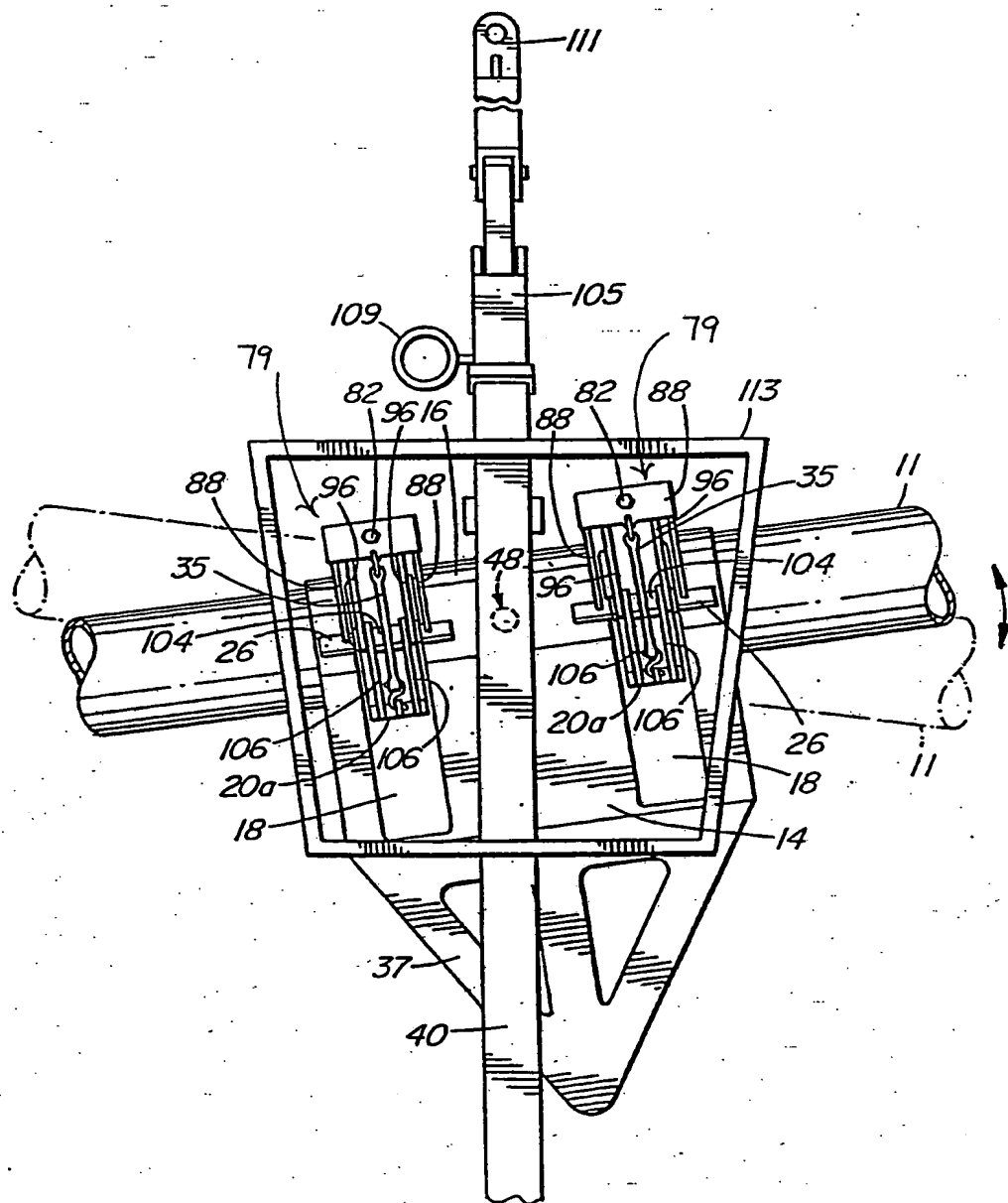


FIG. 12

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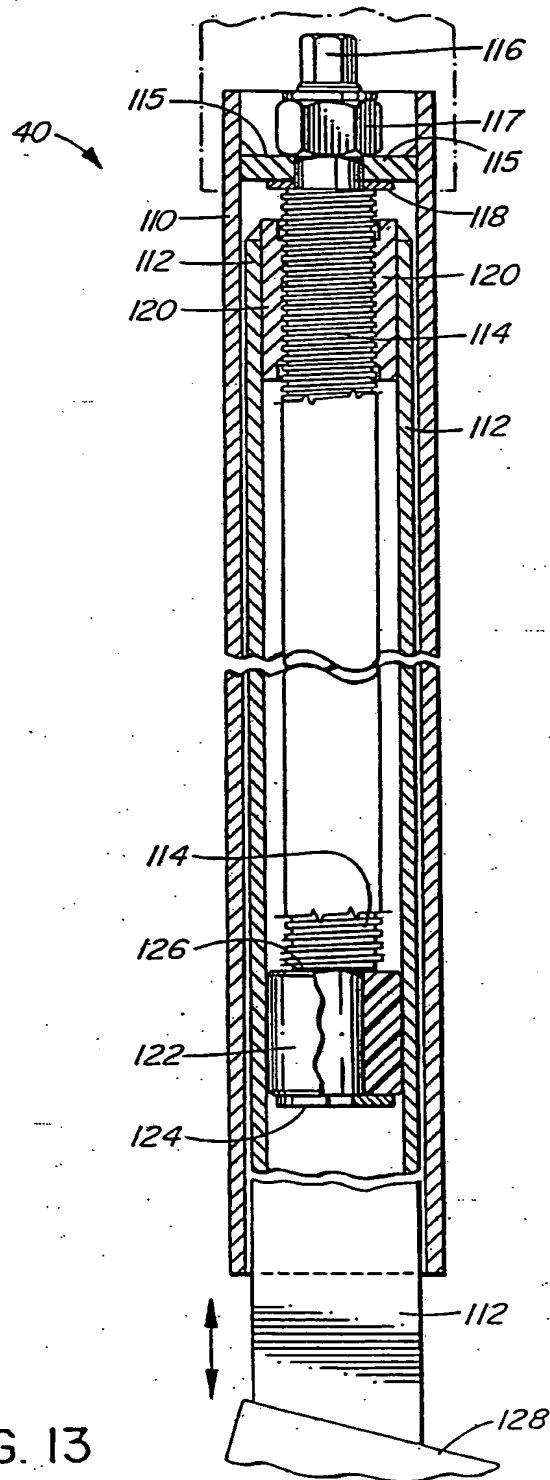
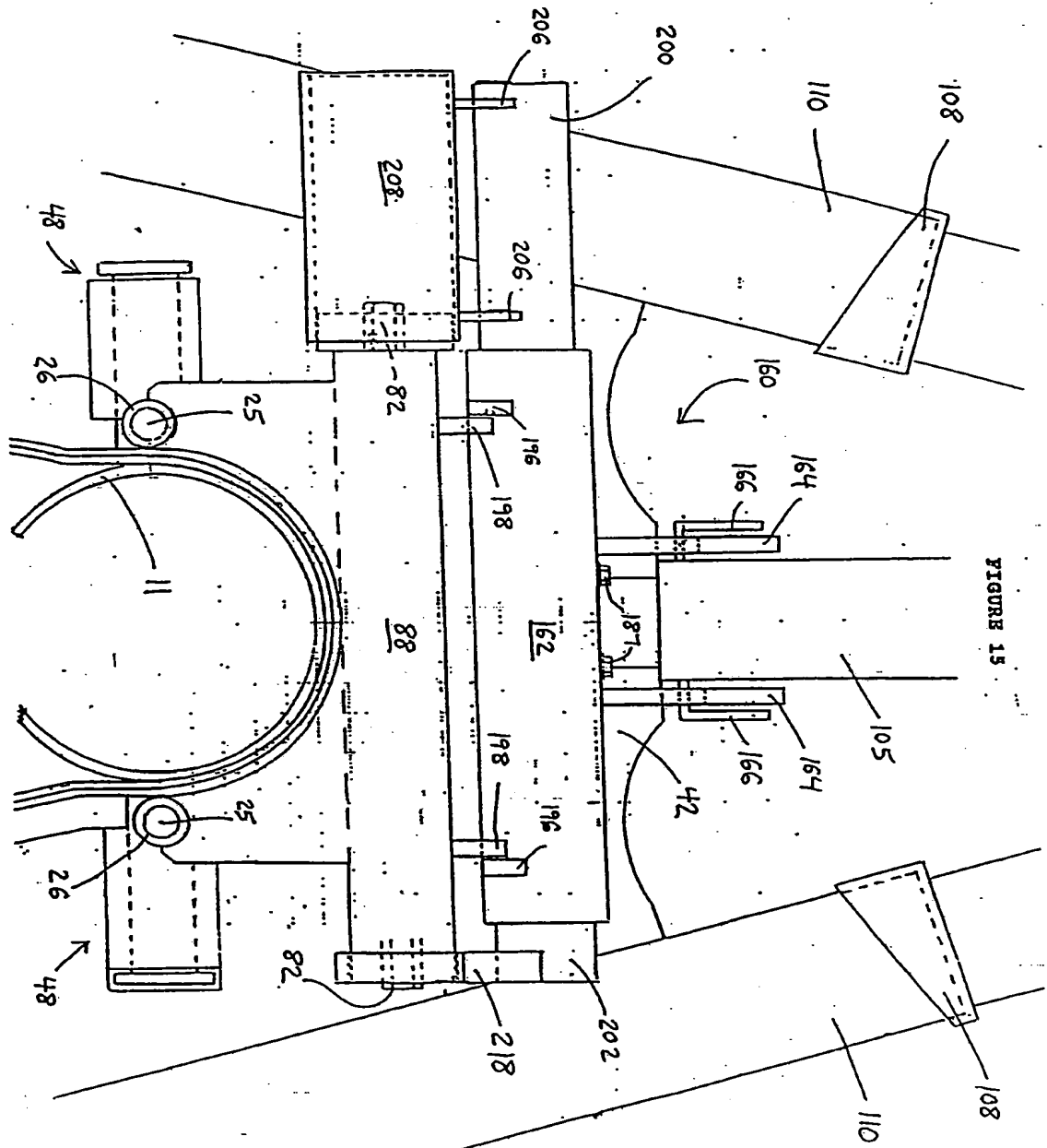


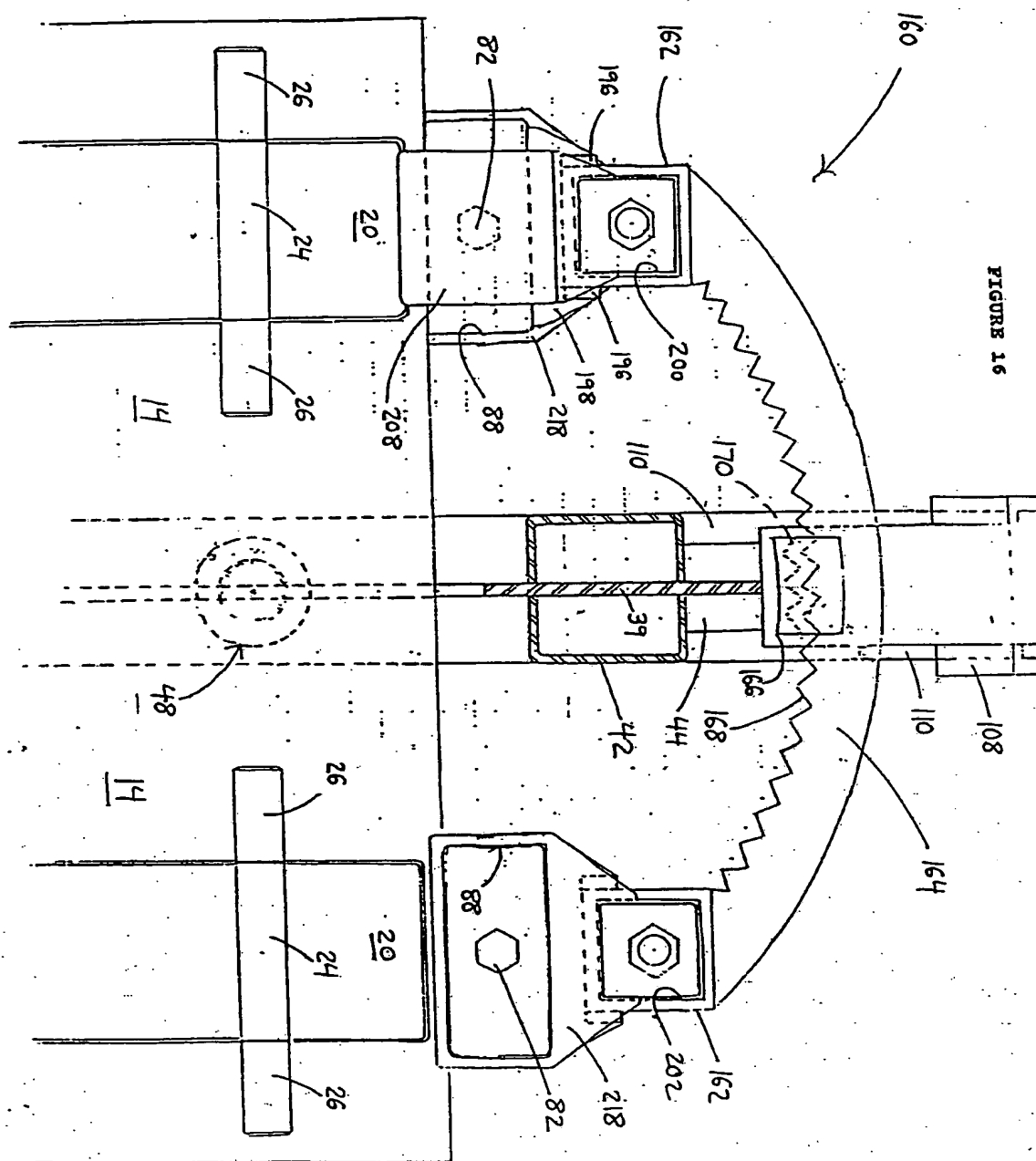
FIG. 13

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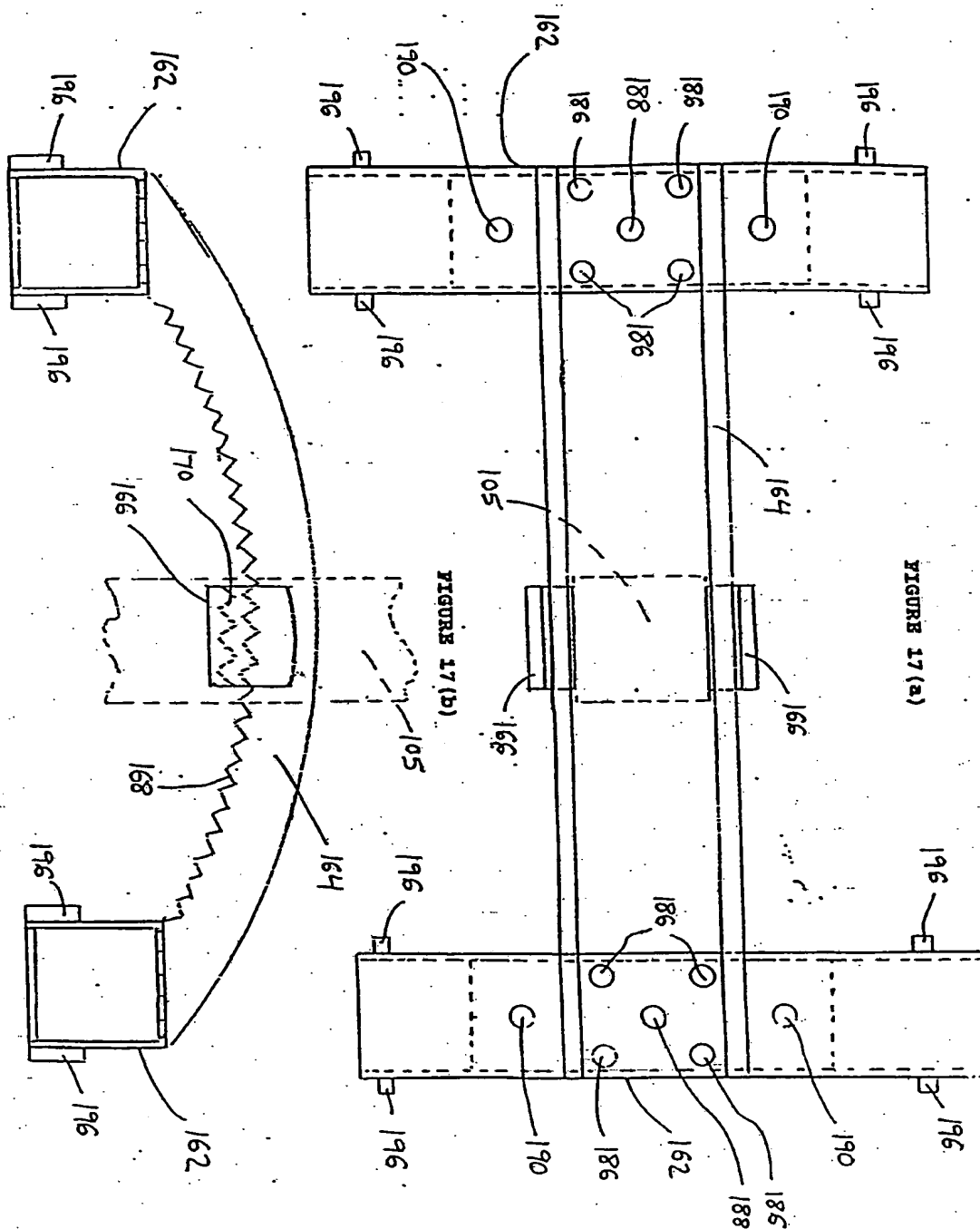


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FIGURE 16



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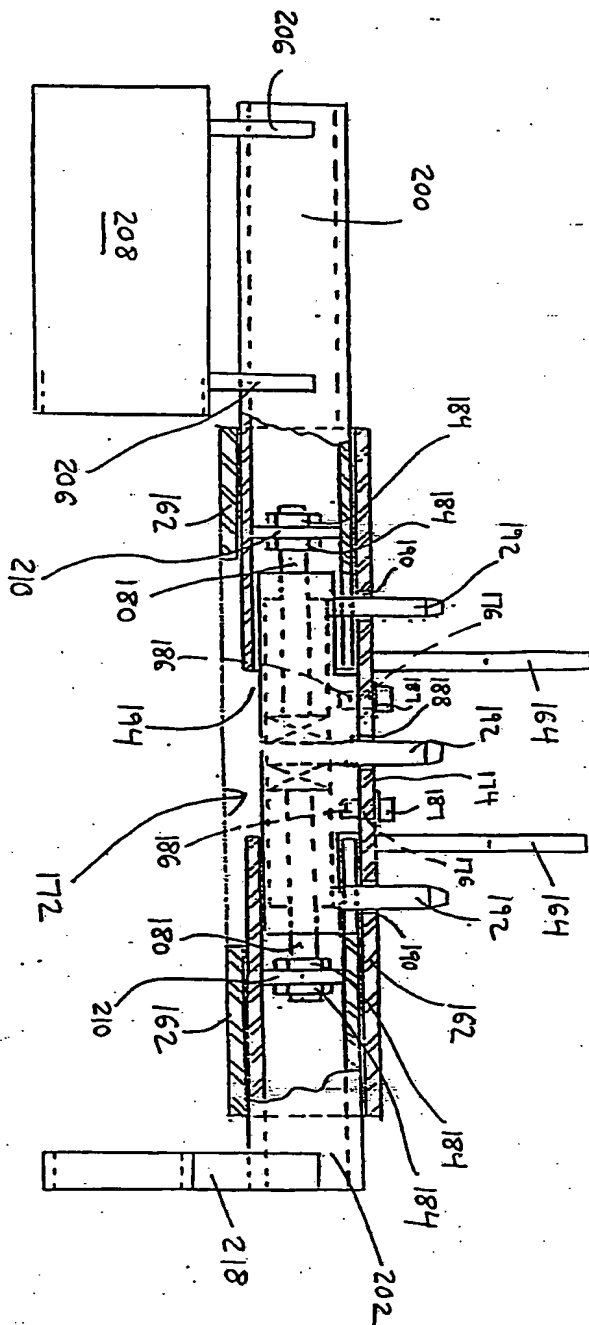


FIGURE 18

FIGURE 19 (a)

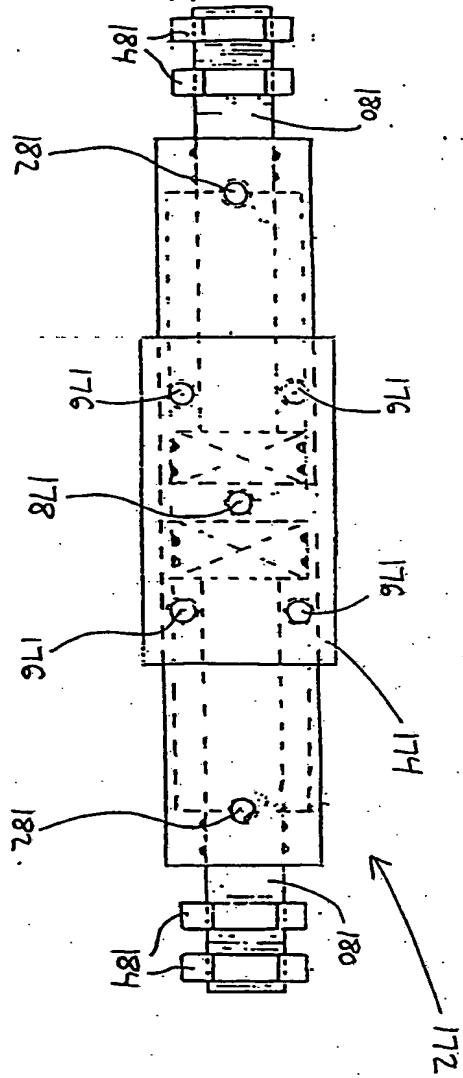


FIGURE 19 (b)

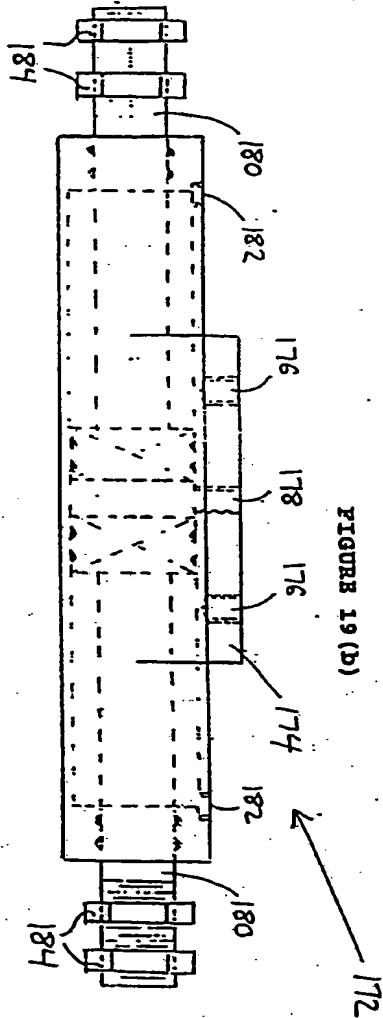


FIGURE 19 (c)

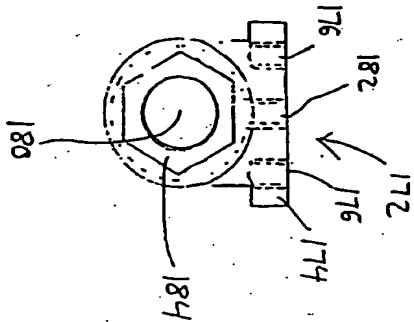


FIGURE 20 (a)

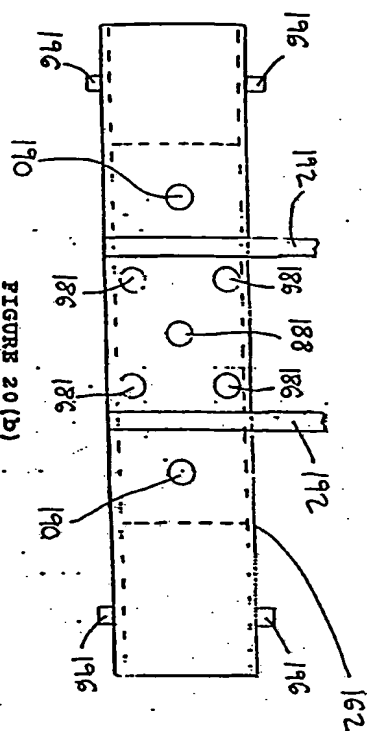


FIGURE 20 (b)

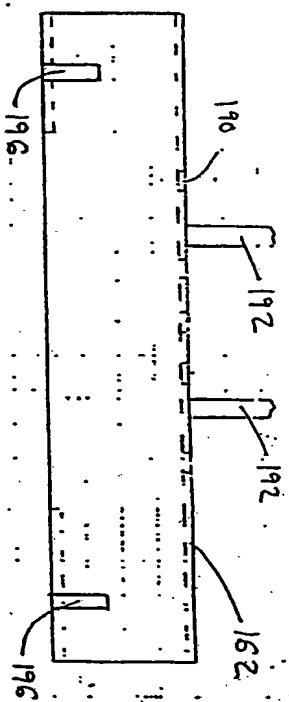


FIGURE 20 (c)

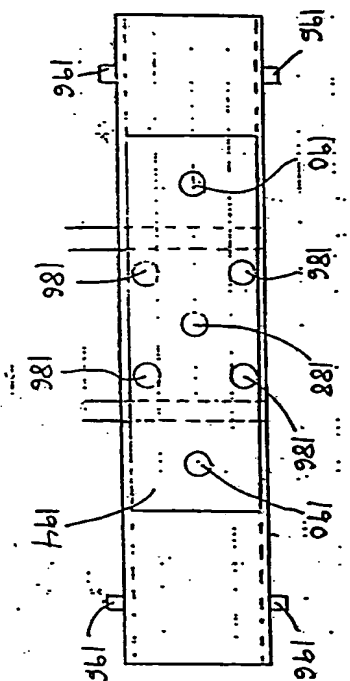


FIGURE 21

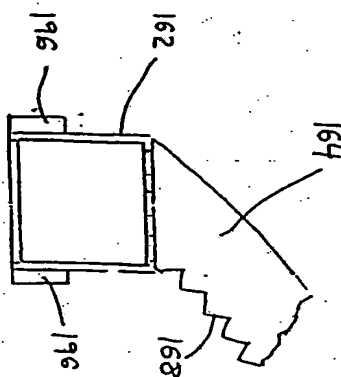
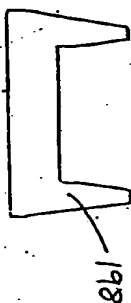


FIGURE 22





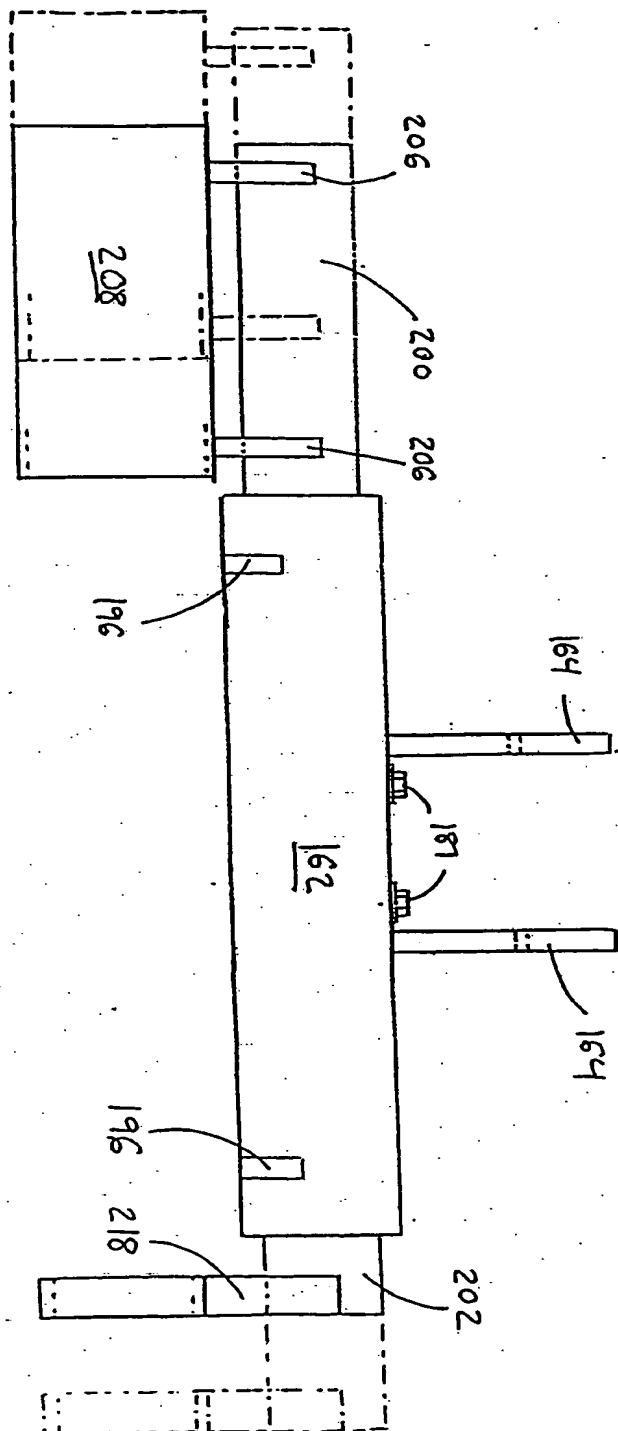


FIGURE 23

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FIGURE 24

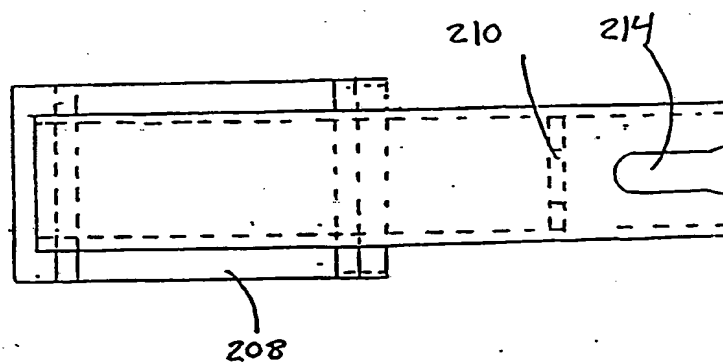
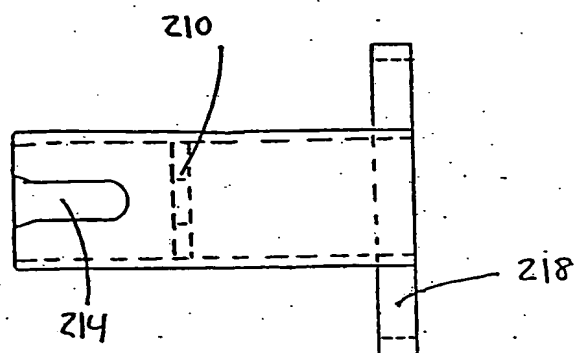


FIGURE 25



## INTERNATIONAL SEARCH REPORT

International Application No

PCT/CA 92/00292

<b>I. CLASSIFICATION OF SUBJECT MATTER</b> (If several classification symbols apply, indicate all) <sup>6</sup>		
According to International Patent Classification (IPC) or to both National Classification and IPC		
Int.Cl. 5 F16L1/20		
<b>II. FIELDS SEARCHED</b>		
Minimum Documentation Searched <sup>7</sup>		
Classification System	Classification Symbols	
Int.Cl. 5	F16L ; B66C	
Documentation Searched other than Minimum Documentation to the extent that such Documents are included in the Fields Searched <sup>8</sup>		
<b>III. DOCUMENTS CONSIDERED TO BE RELEVANT<sup>9</sup></b>		
Category <sup>*</sup>	Citation of Document, <sup>11</sup> with indication, where appropriate, of the relevant passages <sup>12</sup>	Relevant to Claim No. <sup>13</sup>
A	US,A,4 494 893 (MIGLIAVACCA) 22 January 1985 cited in the application see the whole document	-
A	GB,A,2 027 157 (SAIPEM S.P.A.) 13 February 1980 see the whole document	-
A	NL,A,7 317 520 (SAIPEM S.P.A.) 24 June 1974 see the whole document	-
A	US,A,2 226 789 (C.J. TUPY) 31 December 1940 see the whole document	-
-/-		
<p><sup>*</sup> Special categories of cited documents : <sup>10</sup></p> <p>"A" document defining the general state of the art which is not considered to be of particular relevance</p> <p>"E" earlier document but published on or after the international filing date</p> <p>"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)</p> <p>"O" document referring to an oral disclosure, use, exhibition or other means</p> <p>"P" document published prior to the international filing date but later than the priority date claimed</p> <p>"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention</p> <p>"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step</p> <p>"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.</p> <p>"A" document member of the same patent family</p>		
<b>IV. CERTIFICATION</b>		
Date of the Actual Completion of the International Search	Date of Mailing of this International Search Report	
23 OCTOBER 1992	03. 11. 92	
International Searching Authority	Signature of Authorized Officer	
EUROPEAN PATENT OFFICE	BUDTZ-OLSEN A.	

III. DOCUMENTS CONSIDERED TO BE RELEVANT (CONTINUED FROM THE SECOND SHEET)		
Category *	Citation of Document, with indication, where appropriate, of the relevant passages	Relevant to Claim No.
A	US,A,4 624 432 (SALACUSE) 25 November 1986 see the whole document -----	-

**ANNEX TO THE INTERNATIONAL SEARCH REPORT  
ON INTERNATIONAL PATENT APPLICATION NO. CA 9200292  
SA 62116**

This annex lists the patent family members relating to the patent documents cited in the above-mentioned international search report. The members are as contained in the European Patent Office EDP file on  
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